

CFD design and mock up test for heat removal using cylindrical rods mounted on a vertical plate

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▶ Need for high capacitive casks

- ◆ High burn up + short cooling time (60 GWd/tu, 5%enrichment, 2 years cooling time)
- ◆ Consequence : High thermal load up to 6 kW/assembly ->70 kW heat load for 12 spent fuel assemblies in a cask

▶ Maximum thermal power may sometimes be imposed by cooling performance with unexpected orientation of fins after accidental conditions

▶ Need for improved performance of heat convection

- ◆ In normal transport conditions
- ◆ In any configuration (horizontal / vertical)
 - Transport / storage
 - Accidental configuration after 9m drop

▶ What we know about cask fin surfaces

- ◆ Axial fin shape : good for storage / not optimal in transport
- ◆ Radial or annular fin shapes : good for transport / not optimal for vertical cask

Our innovative idea: Rod shaped fins

► Main requirements

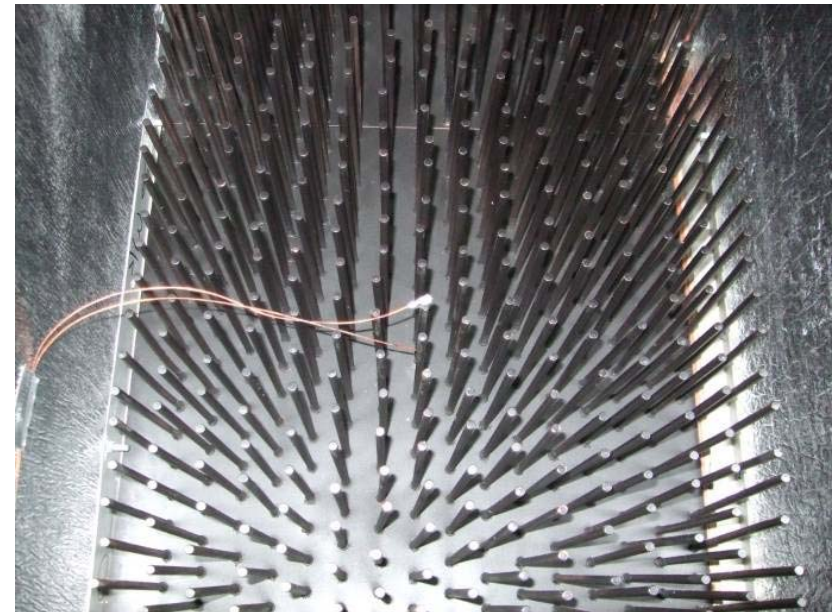
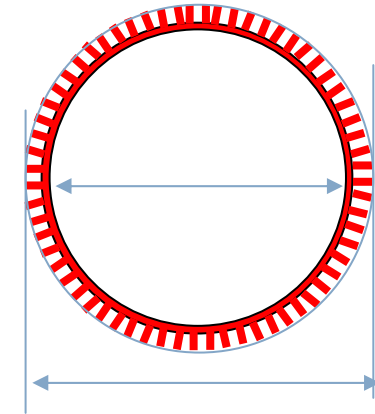
- ◆ Maximum diameter size available (defines max rod length)
- ◆ Use standard diameter and material of rods

► Main benefits are

- ◆ Adjustable increase of cask surface (depending on number and length of rods)
- ◆ Relative independence of cask orientation : to be verified

► Main difficulty is

- ◆ How to optimize the arrangement of the rods in the given space to get the best performance satisfying all the criteria?
 - CFD (computational fluid dynamic) optimization
 - Mock up and thermal tests validation



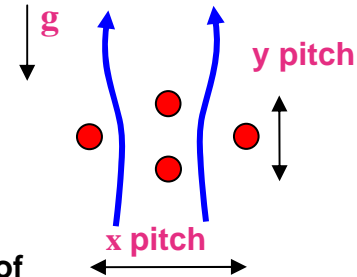
CFD design

CFD model

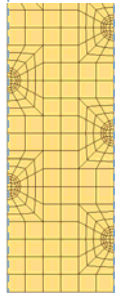
► 3D Model is built with a unit cell (x pitch, y pitch) limited by two symmetric conditions and repeated in the g direction to get the correct overall height (2m)

- ◆ Used of classic turbulent model K/ϵ with high Reynolds law functions
- ◆ Imposed heat (W/m^2) representative of the cask load at the back of the plate / ambient temperature fixed far from plate
- ◆ Main result concerns average plate temperature: The lower the averaged plate temperature is, the better the configuration is.
- ◆ Performance = coefficient ratio A/A^* : improved heat convection coefficient relative to flat plate with no fins. The best configuration is defined by the highest coefficient ratio A/A^*

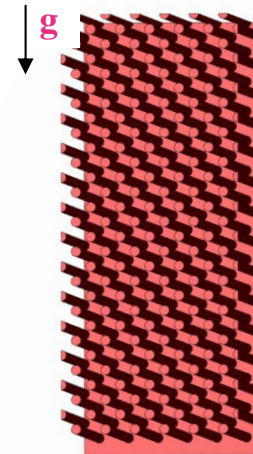
► Calculations are carried out for defined y pitch with various x pitch -> graph for tracking maximum performance in the fixed volume



unit cell for one orientation and mesh

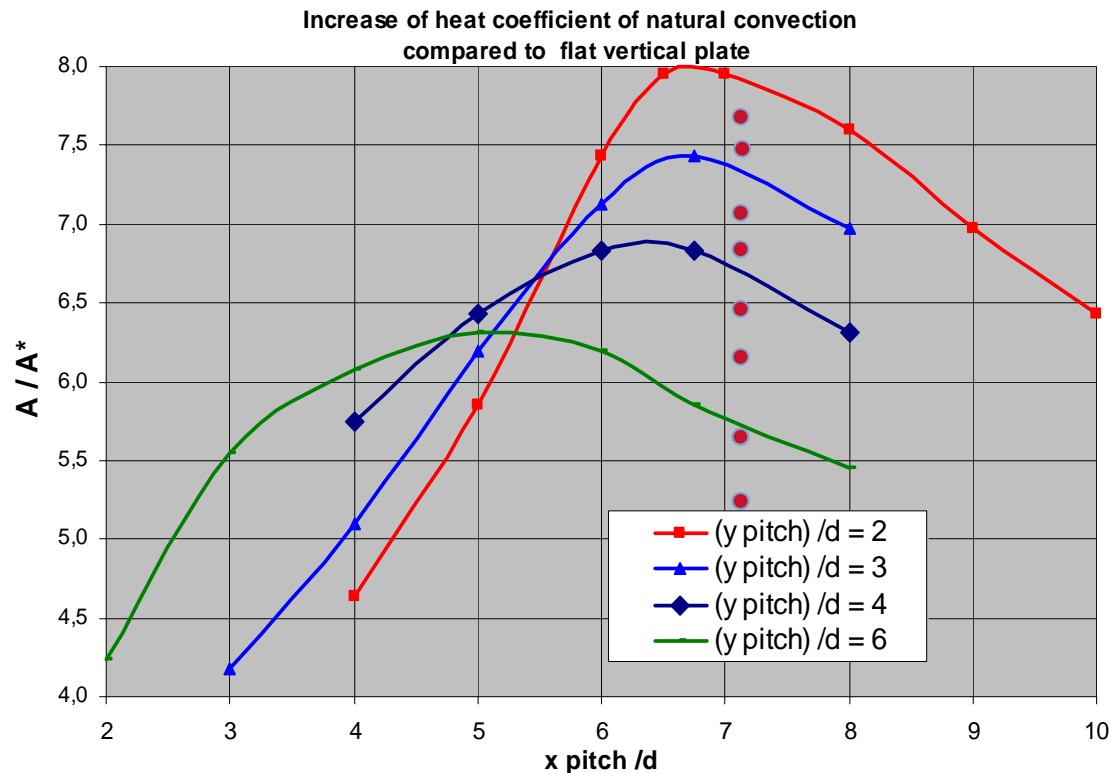


Front view of local mesh



View of extended 3d model (solid part)

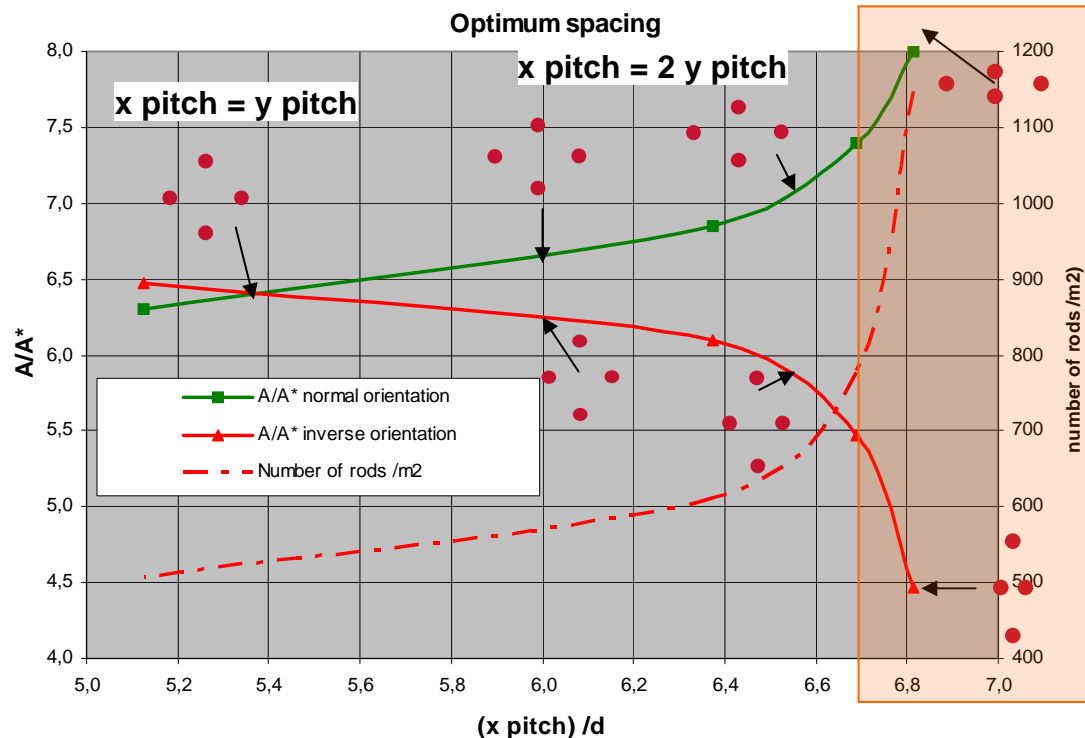
CFD model: calculation result



Key results

- ◆ Each y pitch corresponds to an x pitch optimum which maximizes A/A^* (known result)
- ◆ Maximum performance increases with y pitch reduction but for larger x pitch
- ◆ Performance to flat plate may be increased about 7 to 8 times

CFD model: calculation result



► Optimal path

- ◆ Each gain in maximizing one orientation over performance for $x = y$ pitch will lead to an almost identical same loss in the inverse orientation
- ◆ Rod quantity increases rapidly when $y/d < 3.2$ and $x/d \text{ opt} > 6.7$

Conclusion of CFD design

- ▶ **Maximum performance can be increased compared to reference case where $x = y$ pitch ($A/A^*_{\max} = 6$) but it will come at a cost of a reduced performance in the inversed orientation**
- ▶ **The choice of x / y pitch must be consistent with both criteria in normal and inverse orientation**
- ▶ **x pitch = 2 y pitch is a good compromise between performance in normal and inverse orientation and number of rods**

Thermal tests

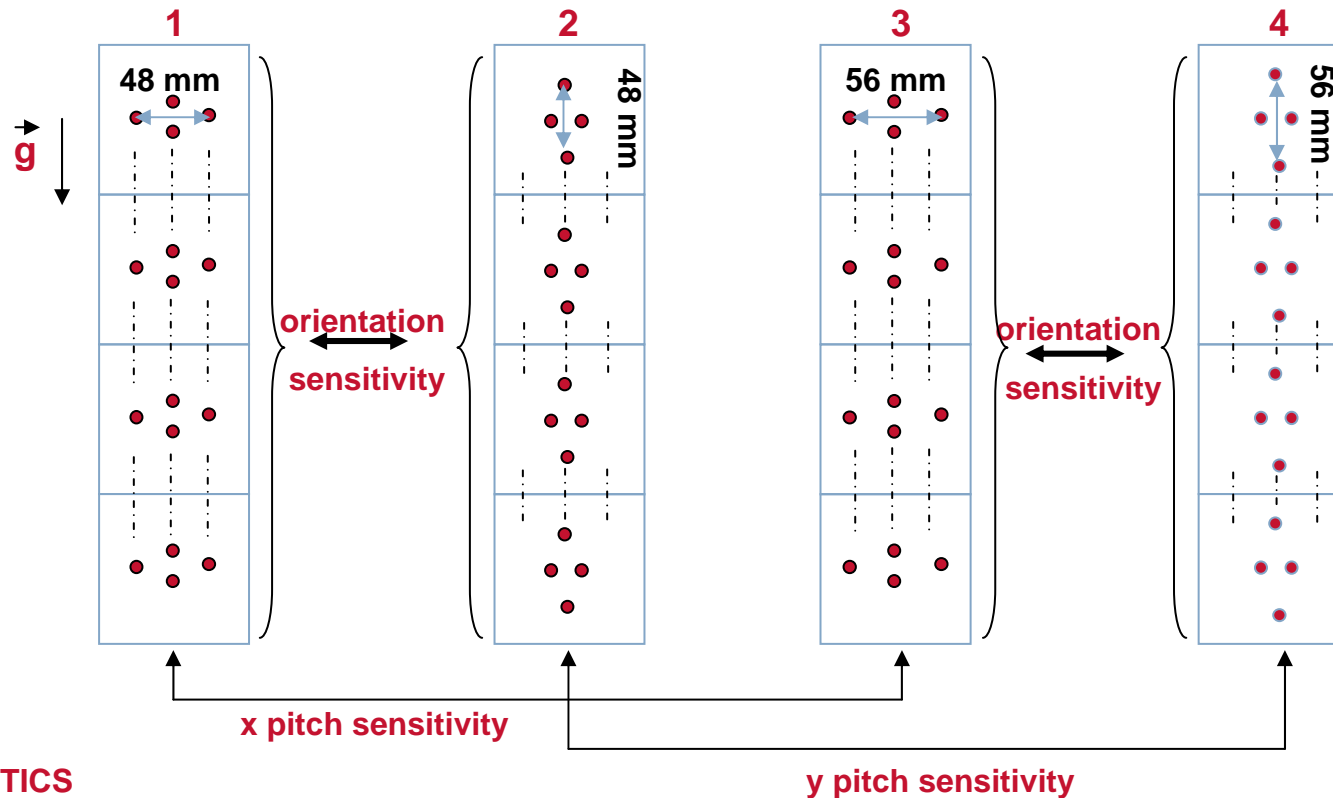
Configuration tests and objectives

► Configuration tests:

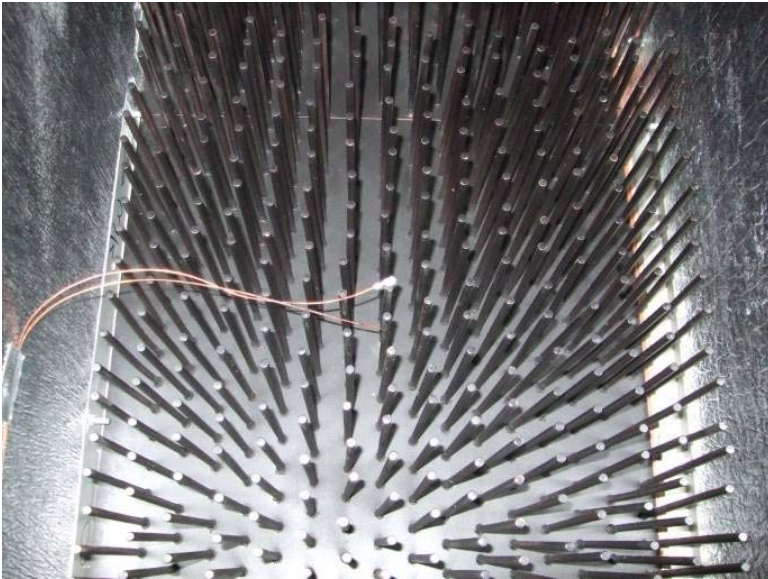
- ◆ Close to optimal calculated with x pitch ~ 2 y pitch (48 mm or 56 mm and y pitch = 24 mm)

► Main objectives:

- ◆ Check that required performance is reached
- ◆ Comparison to calculation : check the predictability of CFD calculation for variation of performance relative to pitch sizes and orientation



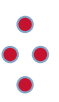



Test mock up



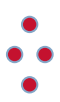



- ▶ **Rods fins welded on a vertical plate (2m high / 0.5m wide) /black coated with a known high emissivity**
- ▶ **20 thermocouples / 2 ambient temperatures / infrared thermography**
- ▶ **Thermal power controlled by four flexible heating mats (stuck to back of the plate) and insulated from ambient thermal losses**
- ▶ **Lateral foil deflector to prevent lateral aspiration flow**

Calculation compared to test results

Calculation results

A/A ^{***} : increase performance compared to vertical flat plate *based on T _{averaged}	y pitch/d			
x pitch/d		4	8	9.3
	4		6.5	6.3
	8	7.5		
	9.3	7.1		

Test results

A/A ^{***} : increase performance compared to vertical flat plate *based on T _{averaged}	y pitch/d			
x pitch/d		4	8	9.3
	4		5.7	5.5
	8	6.4		
	9.3	6.1		

- ▶ Over estimate of about 15% of performance compared to test (but the objective performance is reached)
- ▶ Almost same variation of performance with y pitch (-3% (calc) / -3.5%(exp)), and with x pitch (-5%, -4.5%)
- ▶ The orientation change is correctly predicted (-13%(calc) / -10%(exp))
- ▶ Gap between calculation and test is acceptable
- ▶ Tests confirm the predictability of the calculation and the precision of the absolute value with less than -15%

LOGISTICS

Conclusion

- ▶ **Solution for heat removal in both horizontally and vertically orientated casks using long rod shaped fins has been developed and tested**

- ▶ **CFD helped to define:**
 - ◆ **Optimal arrangement of rods for maximal performance in horizontal configuration while controlling performance in the vertical configuration**
 - ◆ **Design parameters to reach expected performance for both configurations (choice must be consistent with thermal criteria of sensitive components in both NCT and ACT)**

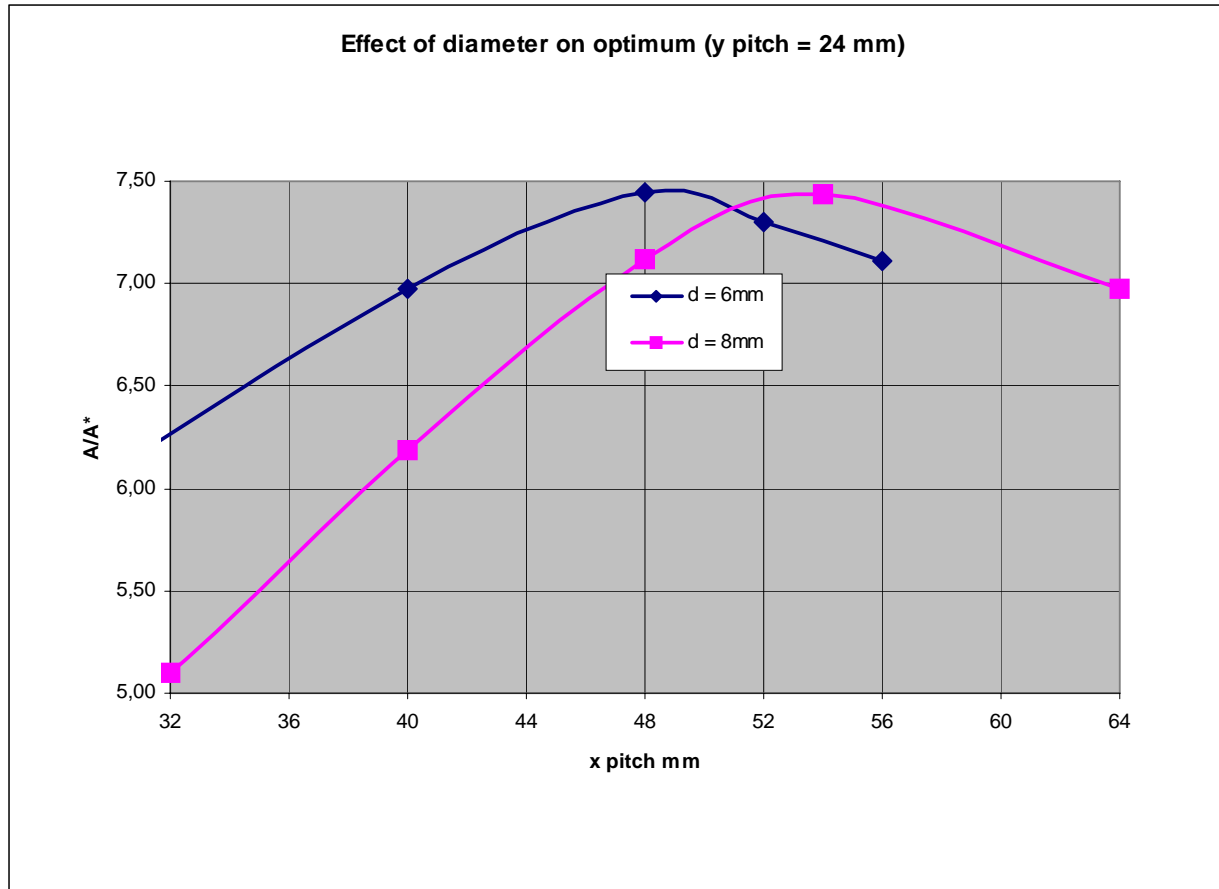
- ▶ **Comparison with mock-up tests shows a good correlation between tests and calculations which validates the concept**

- ▶ **A complementary realistic cylindrical mock-up has been built and tested which confirmed the performance on the plate mock-up (not shown here)**

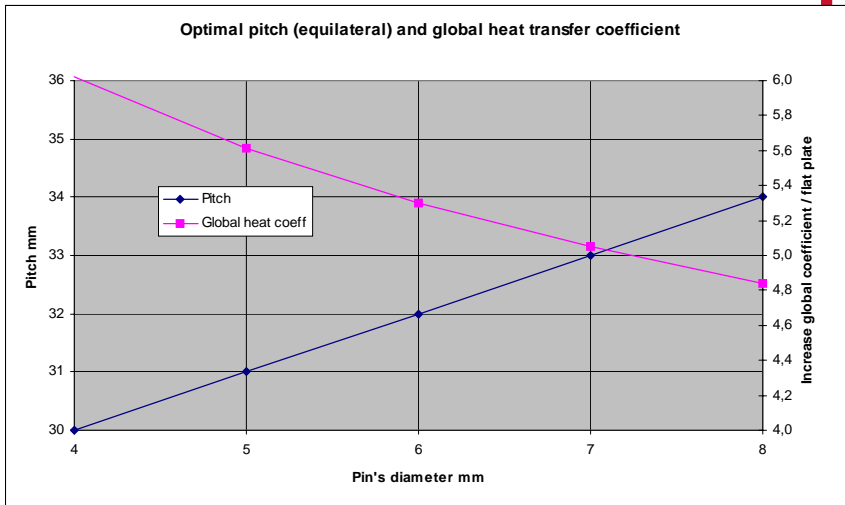
- ▶ **Further investigations should concern industrialization of this innovative solution**

APPENDIX

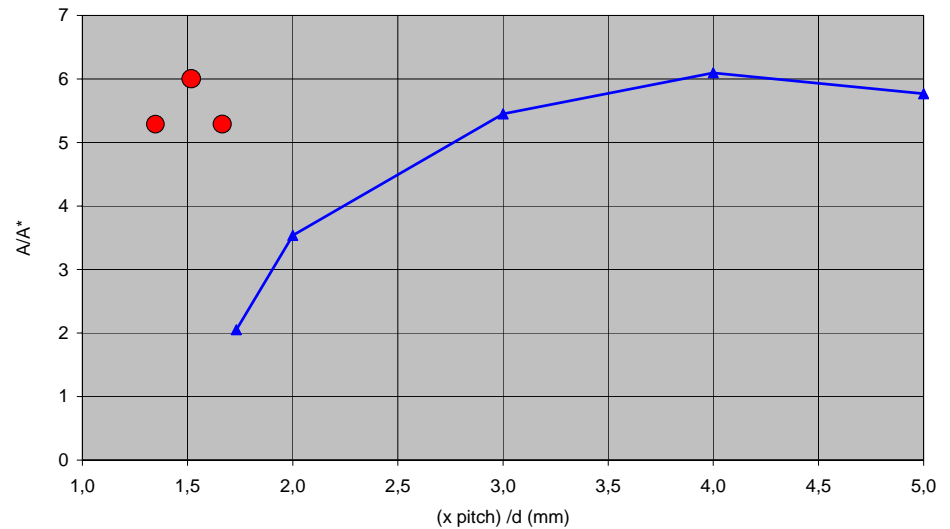
Effect of diameter on optimum



Analytical result for equilateral pitch in laminar flow

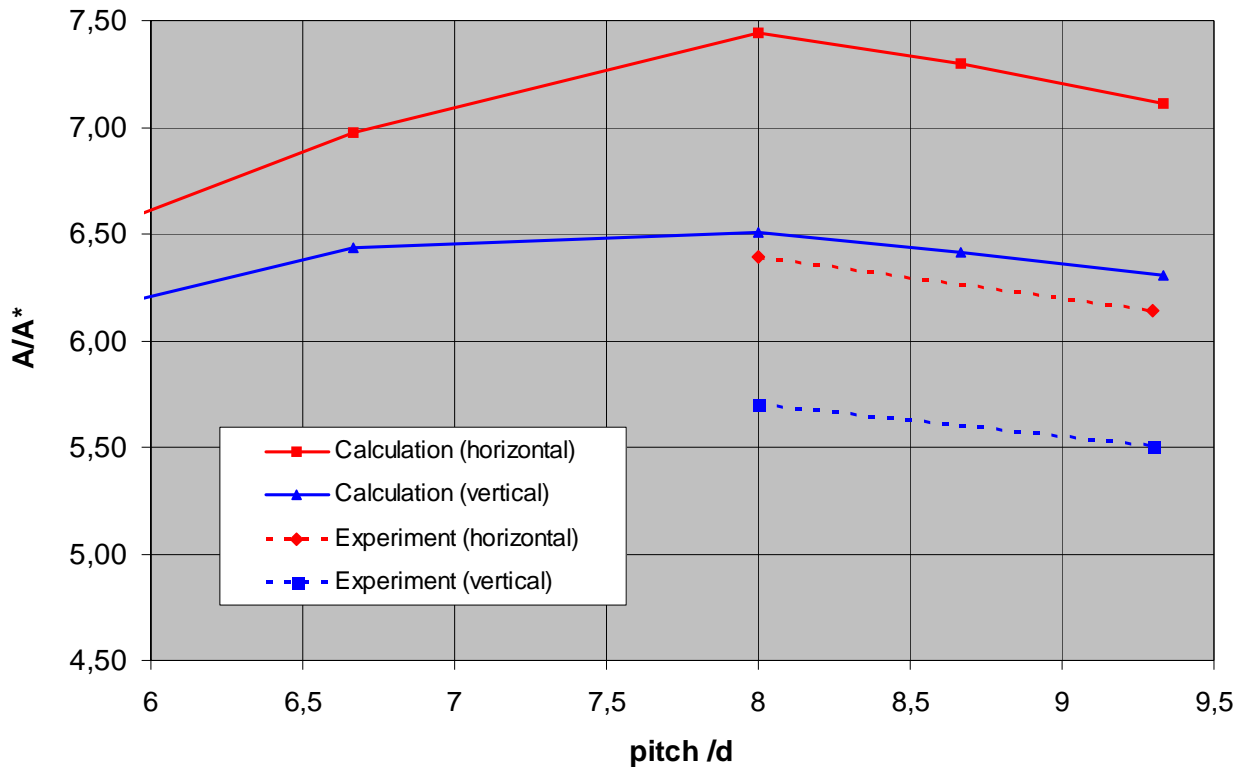


Increase of global heat convection coefficient compared to flat vertical plate for equilateral pitches



Comparison calculation / test

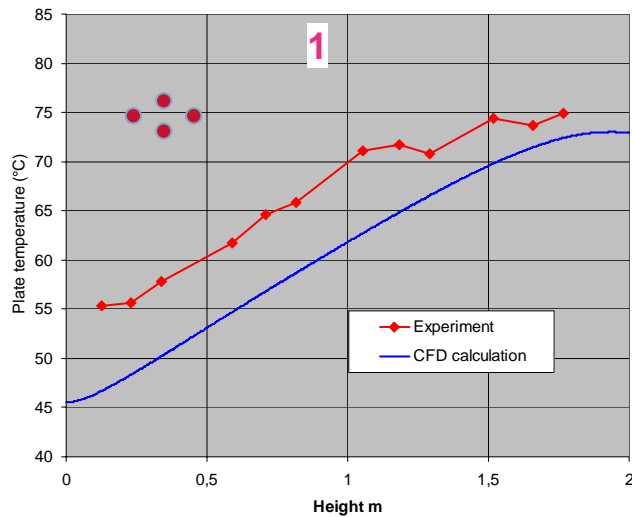
Increase performance of global heat convection coefficient compared to flat vertical plate ($d = 6 \text{ mm}$)



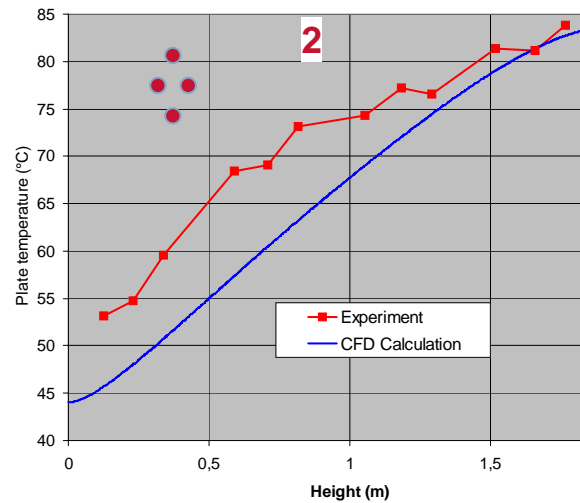
Test results / compared to calculation



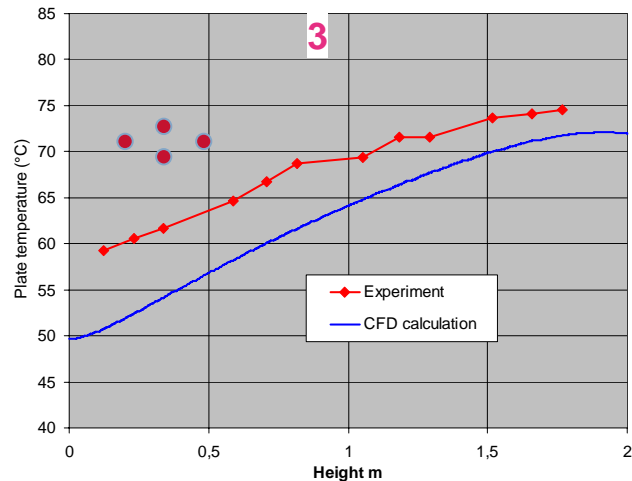
Test 1 : (x pitch)/d = 8 (y pitch)/d = 4



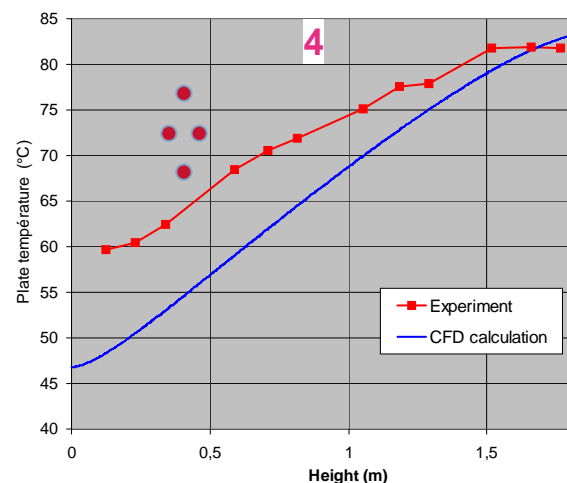
Test 2 : (x pitch)/d = 4 (y pitch)/d = 8



Test 3 : (x pitch)/d = 9.3 (y pitch)/d = 4



Test 4 : (x pitch)/d = 4 (y pitch)/d = 9.3



- ▶ Same trends are observed for both calculation and test even though the effect for calculation is a little bit stronger
- ▶ Under estimation of T calculated
- ▶ Significant increase in T in changing orientation
- ▶ Very slight rise in T in increasing x pitch or y pitch meaning we are close to maximum performance

SCOPE



- ▶ **CONTEXT**
- ▶ **OUR IDEA : ROD SHAPED FINS**
- ▶ **CFD DESIGN**
- ▶ **THERMAL TESTS**
- ▶ **CONCLUSION**