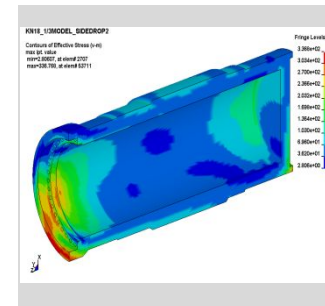


# **Validation of Numerical Simulation Method Using a 1/3-Scale Model Drop Test of KN-18 SNF Transport Cask**



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**K.S. Kim, J.S. Kim, K.S. Choi, I.S. Jeong**

 **Korea Nuclear Engineering & Service Corporation(KONES)**

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# 1. Introduction

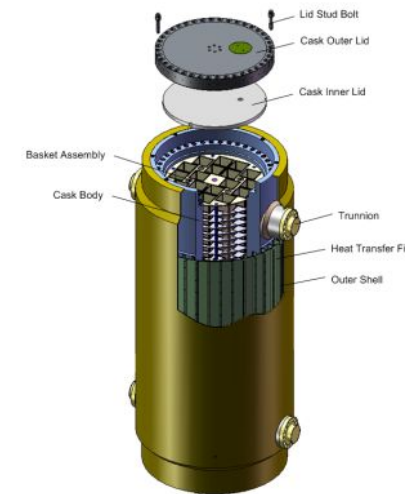
## [Background & Objective]

### ◆ BACKGROUND

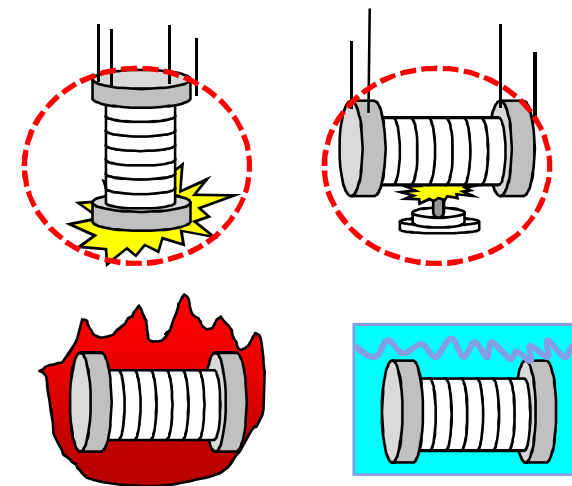
- **KN-18** is a newly developed SNF transport cask in South Korea by KHNP and KONES (for 18 PWR) (▶)
- **Regulations** for SNF transport cask  
:9m drop, 1m puncture, 800°C fire, 200m immersion (▶)
- Two types of method for integrity assessment : Numerical simulation using FE-method, Actual test
- **Structural performance of KN-18** was demonstrated in the SAR by the analysis using state-of-the-art FE-methods via LS-DYNA(another paper)

### ◆ OBJECTIVE

- A series of actual drop tests using 1/3-scale model
- FE-analyses of the scale model cask in all the drop test conditions (same numerical method)
- This paper presents the **dynamic impact characteristics** of the cask from test and analysis results
- The **validation of numerical simulation method** used in the analyses by showing the correlation between test and analysis results



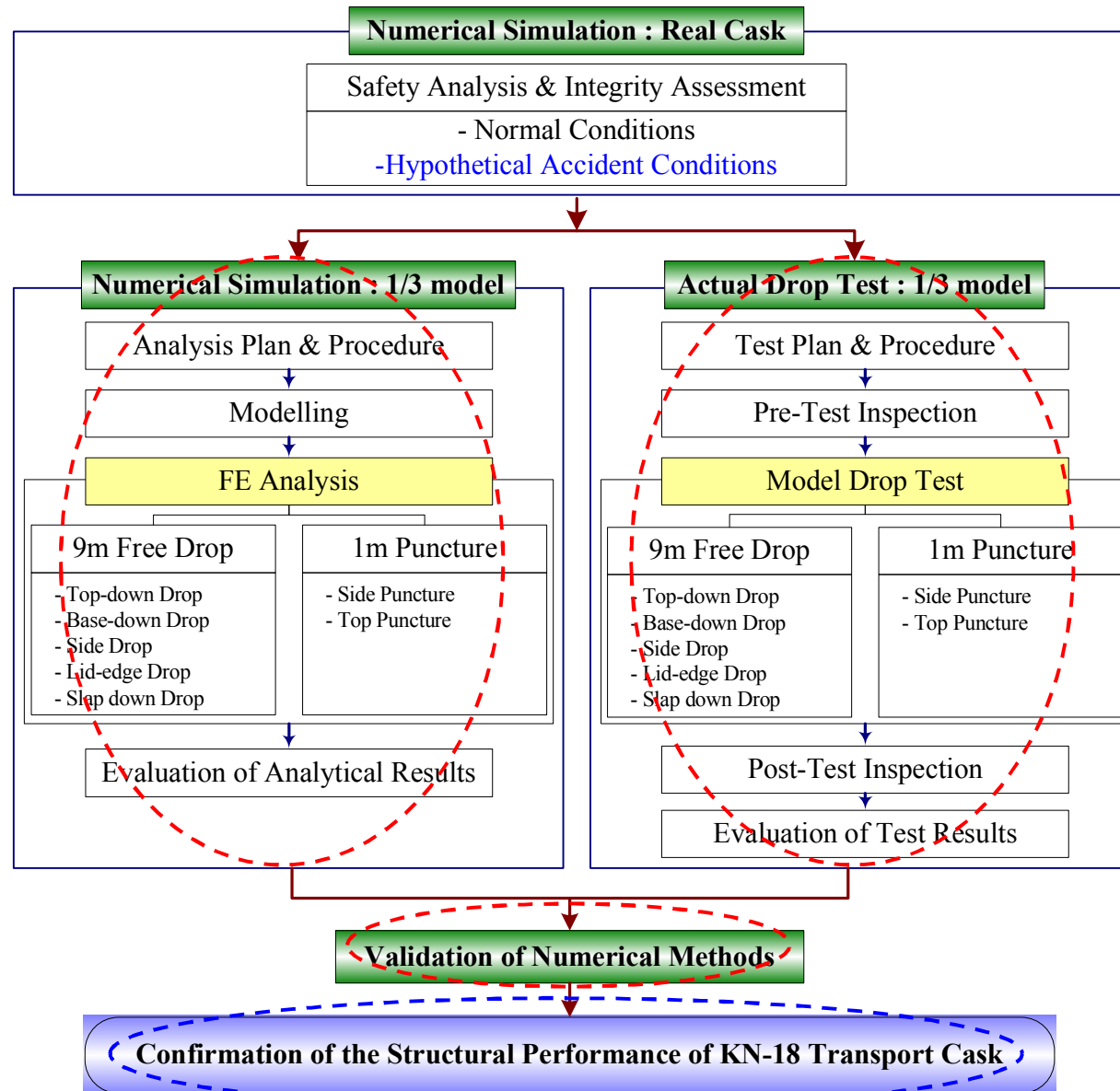
[KN-18 SNF Transport Cask]



[Regulations]

# 1. Introduction

## [Scope & Flowchart]



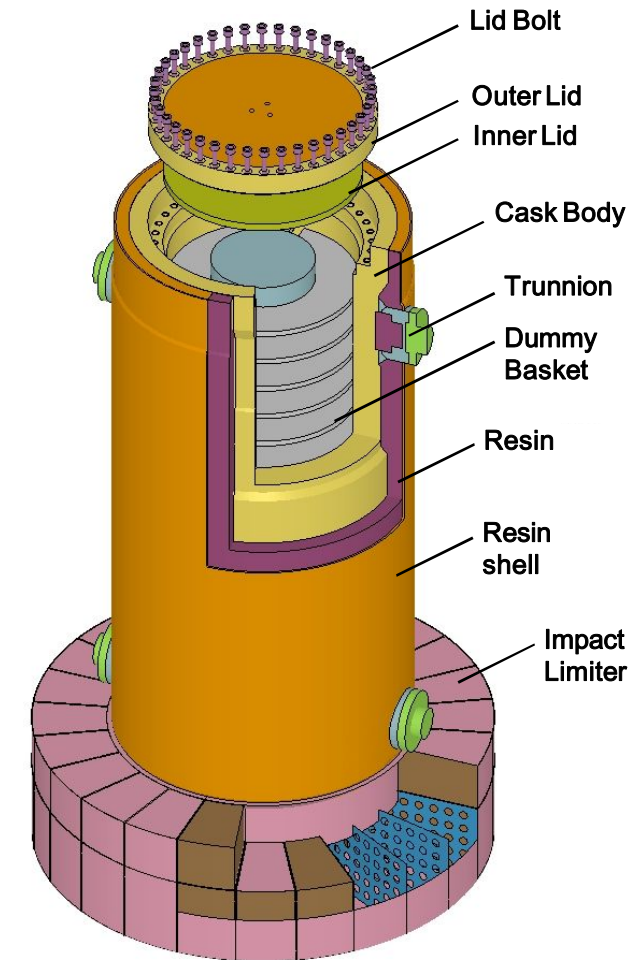
## 2. Cask Model Description

## [Scale model design]

### ◆ Scale Model Design

- **Scale down** original KN-18 **by factor of 3**
- Basically, **same design** as a real cask
- (▷) **Overview** of scale model
- (▷) **Consist of** cask body, outer & inner lid, lid bolt, resin, resin-shell, basket dummy, trunnion(2 set)
- (▷) **2 Impact limiter** : *Beech & spruce, housing, gusset*
- **Overall size** : Length(2.101m), Diameter(0.782m)
- **Total weight** : 4.60 ton (with 2 IL)
- **Material property (steel)** (▽)

Material	A-350 LF3	A-182 GR.F6NM	A-240 TP304	A-453 GR.651
Components	Cask Body	Cask Lid	Resin-shell	Lid Bolt
Elastic Modulus (MPa)	1.91E05	2.01E05	1.95E05	1.95E05
Poisson Ratio	0.31	0.29	0.29	0.29
Yield Stress(MPa)	341.5	765.7	255.0	485.0
Tensile Stress(MPa)	571.1	890.4	609.0	990.0
Density (ton/mm <sup>3</sup> )	7.62E-9	7.81E-9	8.00E-9	8.00E-9
ETAN(Mpa)	1423.6	1594.0	1550.5	3265.1



[1/3-scale model]

# 2. Cask Model Description

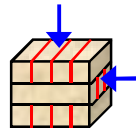
## [Fabrication]

### ◆ FABRICATION

#### ➤ Fabrication process

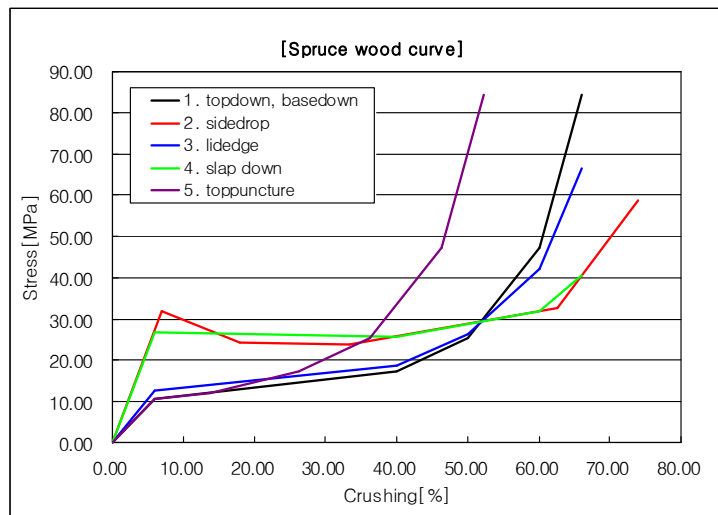
: (▷) cask body->cask lid->resin pouring -> basket dummy->IL housing & gusset-> insertion of wood layer

### ◆ WOOD PROPERTY

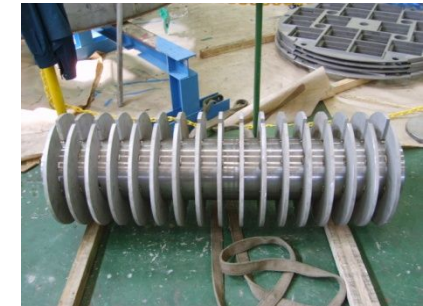


#### ➤ (▽) Wood crush characteristics (FE-input)

- stress-strain behavior(test data)
- MAT\_HONEYCOMB in LS-DYNA



[Wood crush characteristics]



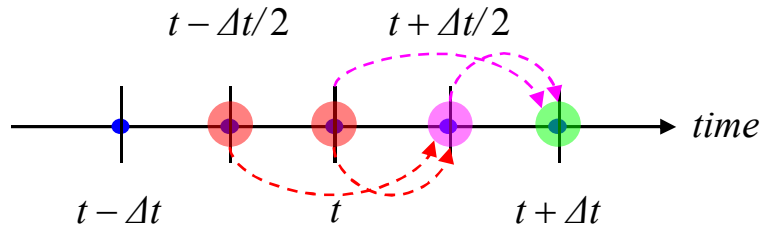
[Fabrication of the test model]

# 3. Outline of Numerical Analysis

## [Analysis method]

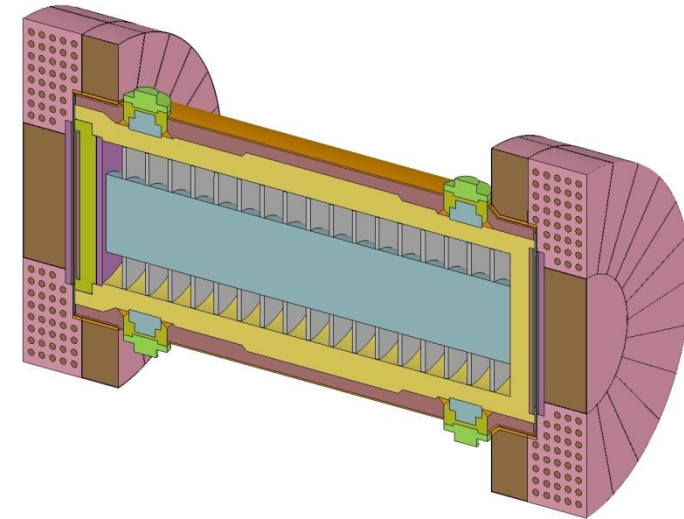
### ◆ ANALYSIS METHOD

- 3D nonlinear dynamic FE-simulation
- FE Code : LS-DYNA explicit v. 970
- Explicit time integration method

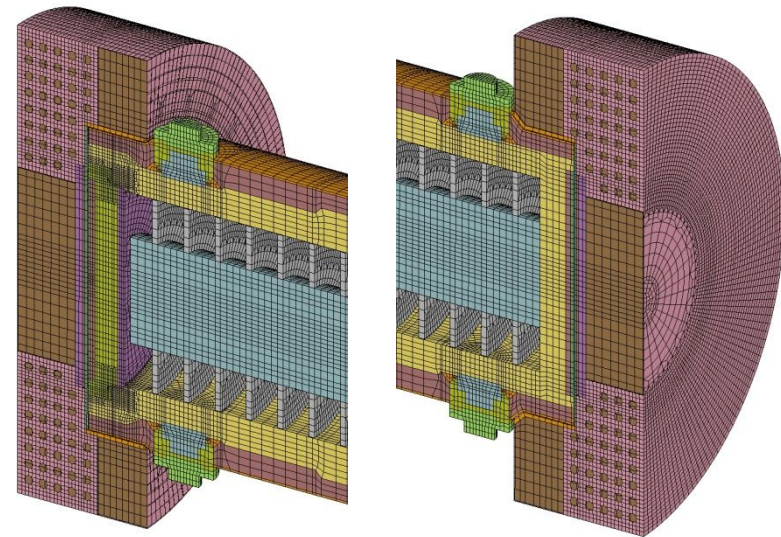


- Velocity and displacements at time  $t + \Delta t$  updated explicitly
- Solution is trivial : diagonal mass matrix, no iteration is required
- Ideal method for high speed impact simulation
- Usually more reliable for problems involving discontinuous nonlinearities (Contact, Impact)
- Limited time increment size -> Automatic

$$\Delta t_{\min} \leq \frac{2}{\omega_{\max}} (\sqrt{1 + \xi^2} - \xi)$$



(▷)[Overall view of the FE-model]



[Mesh of the FE model]

# 3. Outline of Numerical Analysis

[FE-modeling]

## ◆ FE-MODELING

### ➤ 3D detailed FE-model

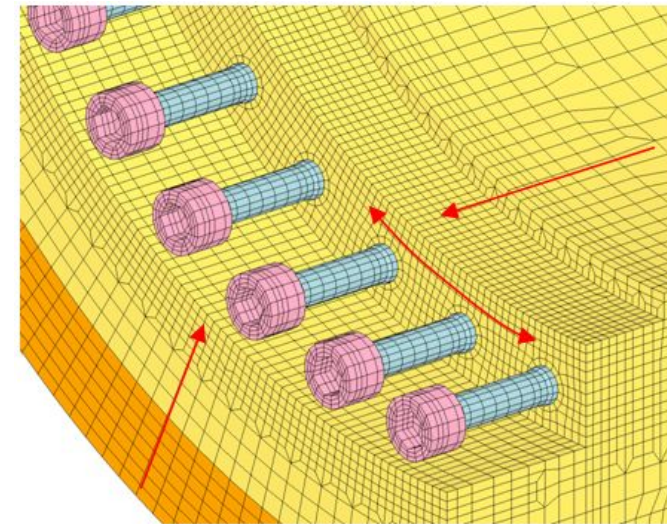
- All components are modeled explicitly
- Only hexahedral solid, rectangular shell

### ➤ Basic half symmetry model

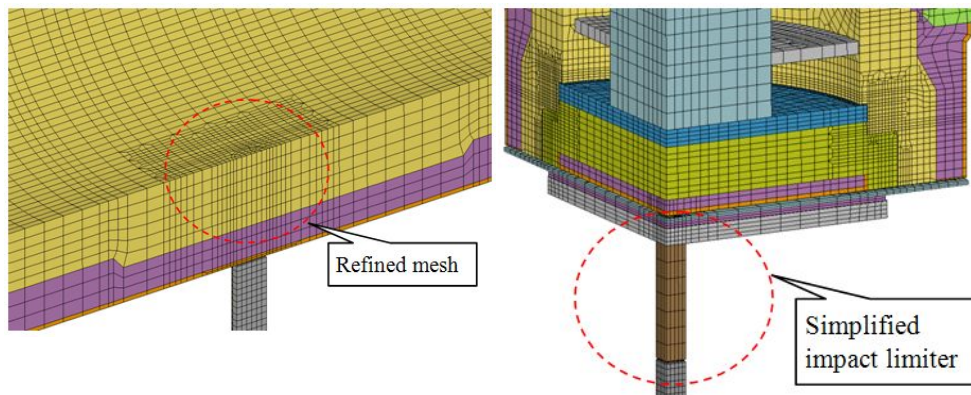
: 445,506 nodes, 369,182 elements

### ➤ Mesh density design

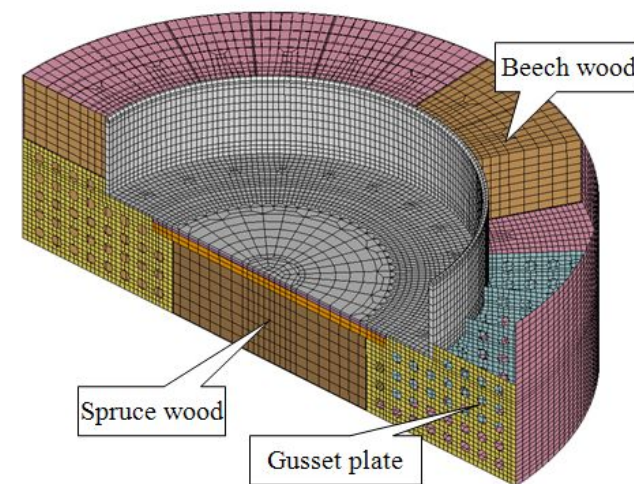
- : Purpose of analysis, package behavior
- Fully integration solid element : Thin wall
- Puncture model : modification of basic model
- Material behavior : tri-linear elasto-plastic (steel)



(▷)[Details at the lid-body interface]



(▷)[Details of the puncture model]



(▷)[Details of the impact limiter]



# 3. Outline of Numerical Analysis

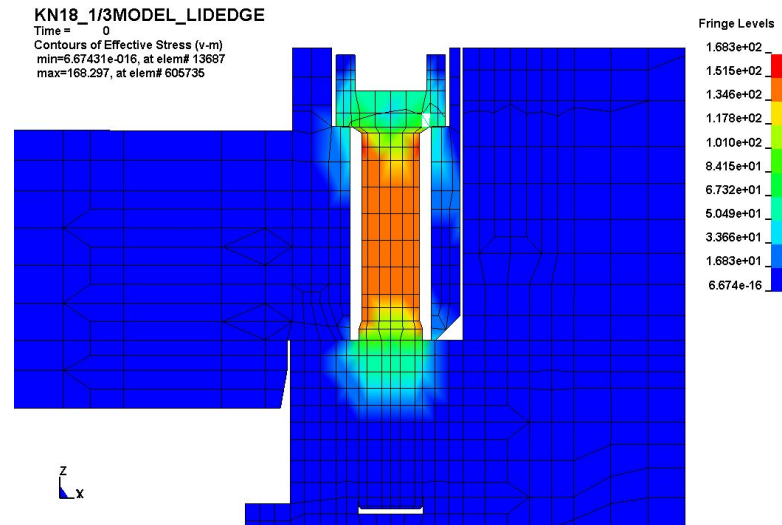
[Analytical assumption]

## ◆ INITIAL & BOUNDARY CONDITION

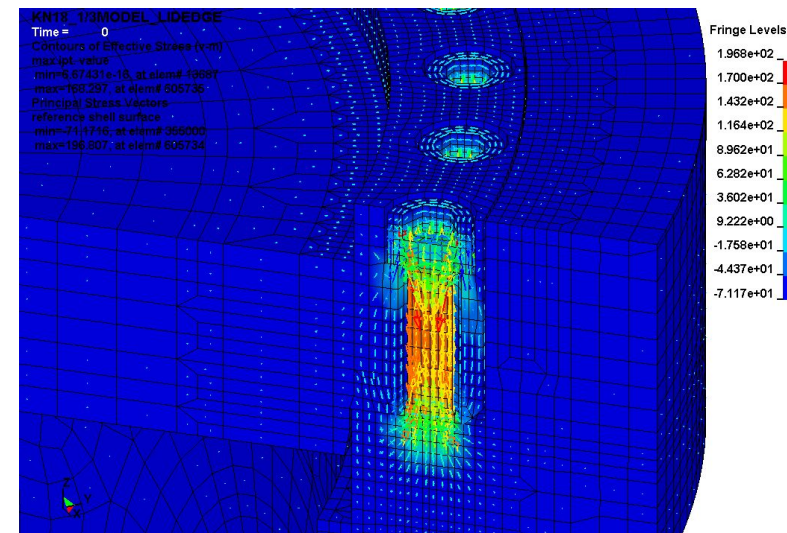
- Two phases analysis
  - Bolt pre force : dynamic relaxation phase
  - Drop analysis : transient phase
  - The stress at the end of the dynamic relaxation phase are carried over to become the initial stress of transient phase (▶)
- Initial velocity : 13.3m/s(9m), 4.43m/s(1m)
  - Model is located close to the target
- B/C : Fix(target , bar), Symmetry condition

## ◆ OTHER ASSUMPTIONS

- Unyielding target : "RIGIDWALL" in LS-DYNA
  - no penetration and no energy absorption
- Nominal dimension in room temperature
- Contact condition : Penalty method(no friction)
- Non impacting IL removed(as in the test model)



[Stress distribution due to bolt pre stress]



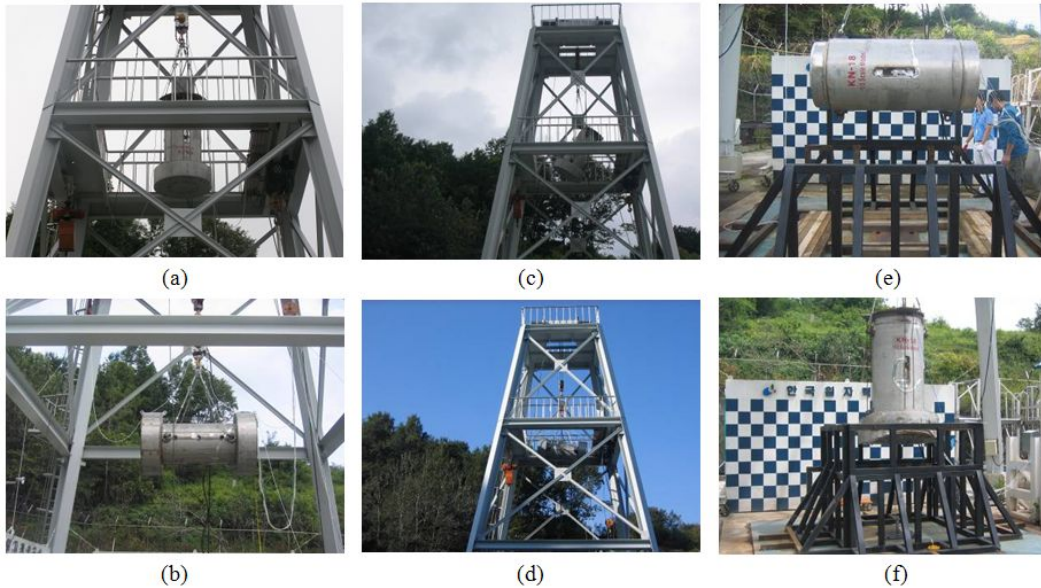
[Principal stress after dynamic relaxation]

# 3. Outline of Numerical Analysis

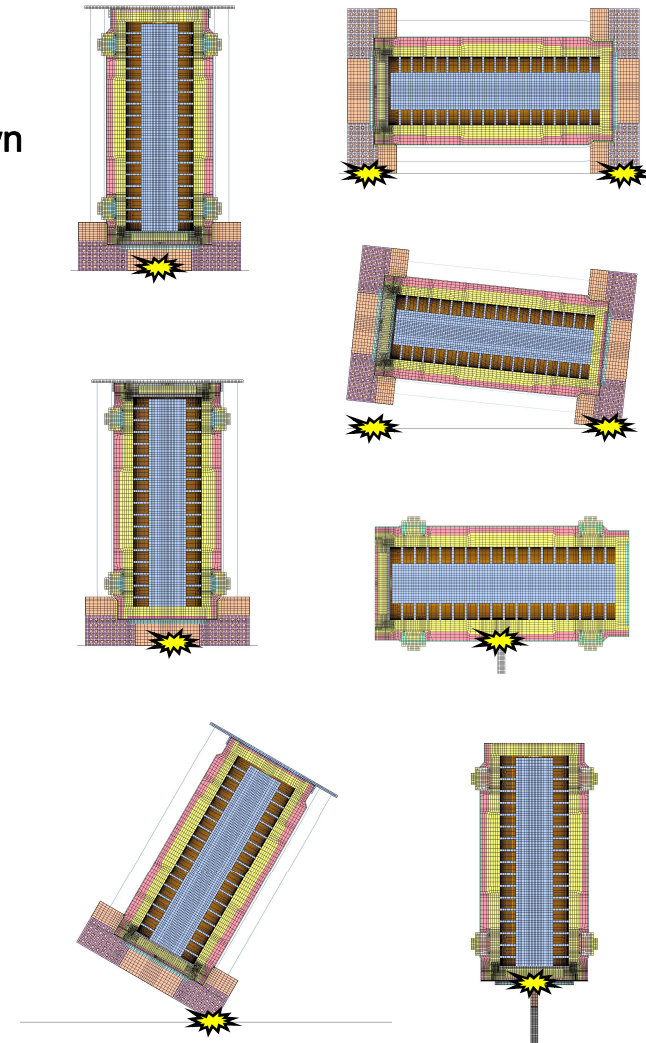
## [Drop orientation]

### ◆ DROP ORIENTATION

- **Total 7case** : to consider worst drop orientation (▷)
  - 9m drop : top-down, base-down, side, lid-edge, slap-down
  - 1m drop : side puncture, top puncture
- **Centrally positioned** at the initial stage
- **CG** is directly over the initial impact point except for slap-down drop case
- (▽) **Initial set-up** of the test model



[Initial set-up : test model]



[Drop orientation : FE-model]

## 4. Outline of 1/3 Scale Model Test

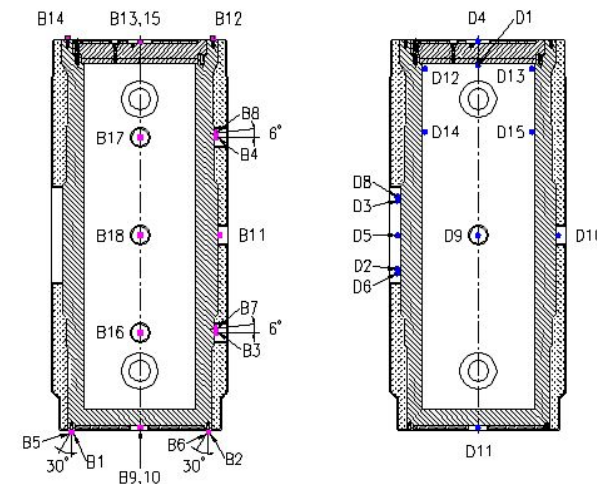
## [Drop test program]

### ◆ TEST AIM

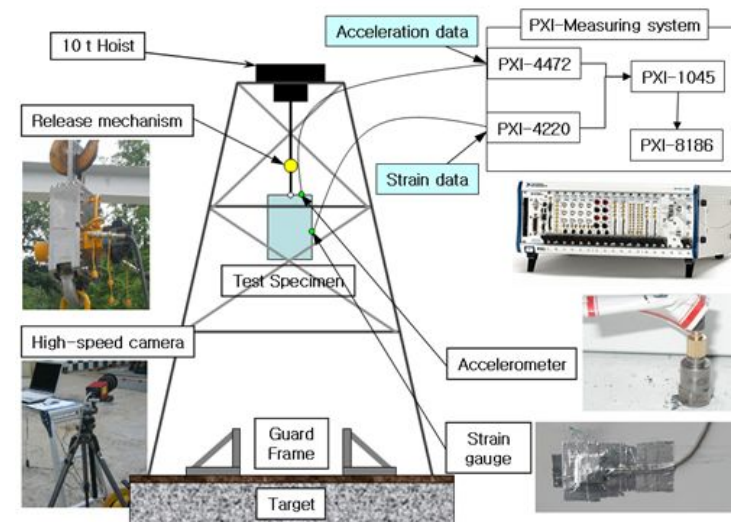
- To confirm the **dynamic impact characteristics** of the KN-18 transport cask
- To **verify** the numerical simulation method used in the analyses

### ◆ TEST FACILITY

- (▷) **Drop test facility**
  - drop tower, unyielding target, 10t hoist, release mechanism, guard frame
- **Measurement system**
  - (▷) **strain gauges, acceleration sensors**
  - high speed camera
  - dynamic data acquisition system
- (▷) **Measurement point**
  - base on the pre-test analysis results
- **Before and after each test**
  - IL deformation, cask dimensions, bolt torque are also measured



[Measurement point]



[Drop test facility and equipment]

# 5. Numerical and Experimental Results

## [Drop behavior]

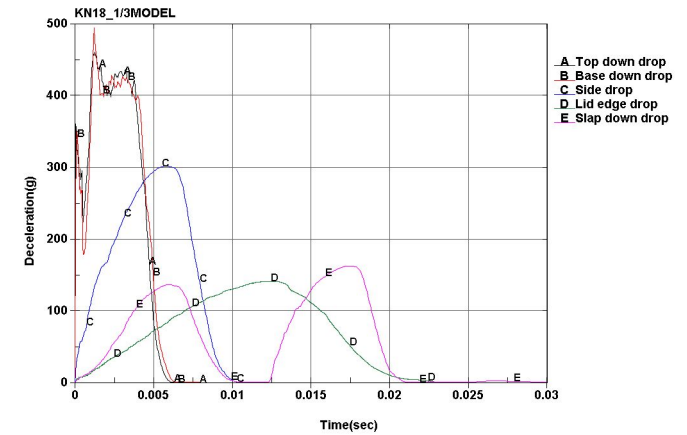
### ◆ OVERALL IMPACT BEHAVIOR

#### ➤ (▽) Evaluation of energy balance

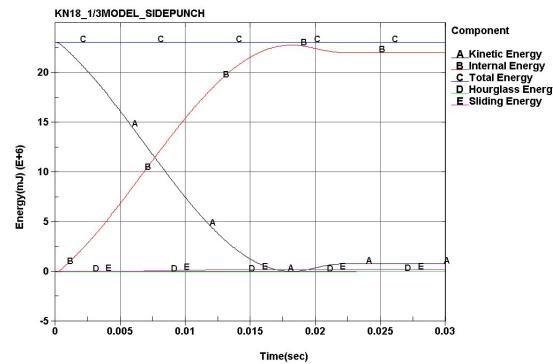
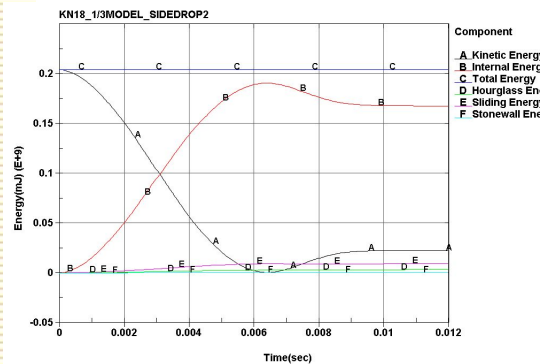
- energy value, smoothness, energy loss, total energy
- it was confirmed that the analyses were performed successfully

#### ➤ (▷) Overall deceleration

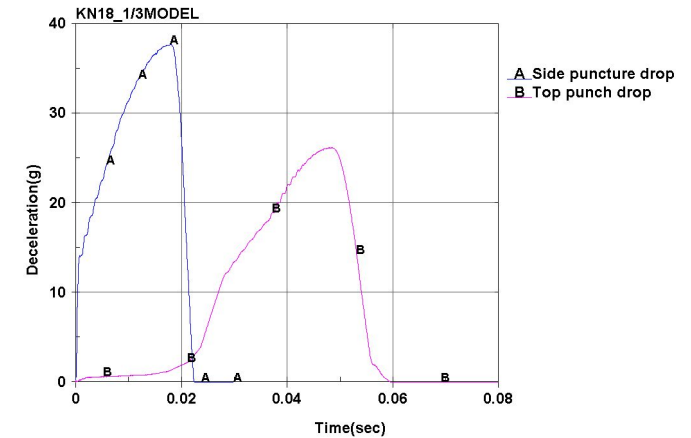
- dividing reaction force by total mass
- max. value (9m): 141g (lid edge) ~ 494g (base-down)
- max. value (1m): 26g (top) ~ 38g (side)
- slap-down : 2 peak, first impact < second impact



[Overall deceleration time-history(9m)]



[Energy time-history]

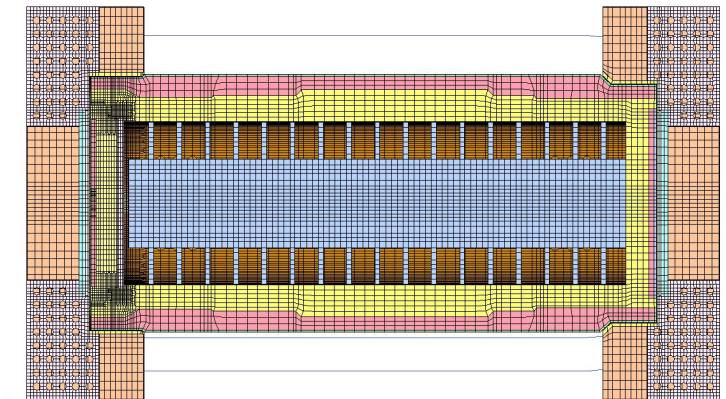


[Overall deceleration time-history(1m)]

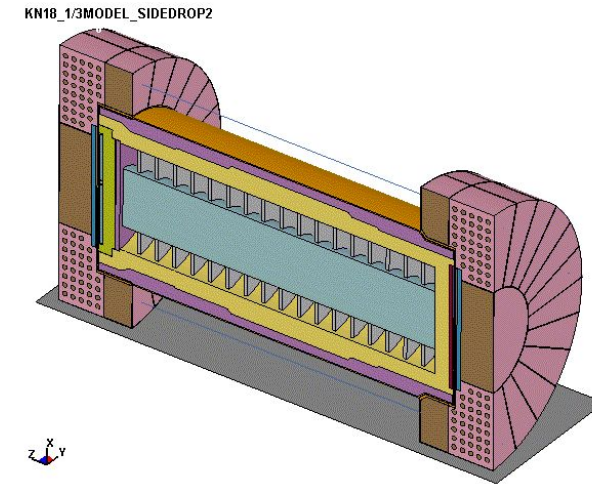
# 5. Numerical and Experimental Results

## [Drop behavior]

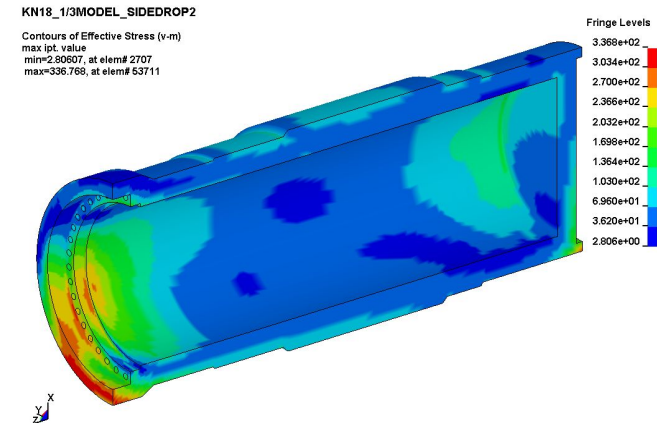
- ◆ **IMPACT BEHAVIOR : 9m side drop**
  - (▽) **Drop orientation** at the initial stage
    - free drop on unyielding target
    - axis horizontal orientation about the target
  - (▷) **Drop simulation** results
  - Cask deflected like a **simple beam** supported at the top and bottom ends
    - *tensile stress on the side closest to target*
    - *compression stress on the opposite side*
  - (▷) **Stress distribution** of the cask body
    - high stress at the top end of the cask was caused by oval deflection behavior



[Drop orientation]



[Drop animation]

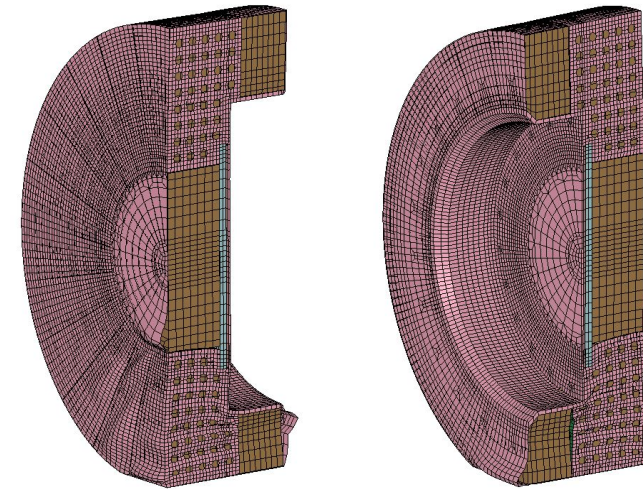


[Stress distribution of the cask body]

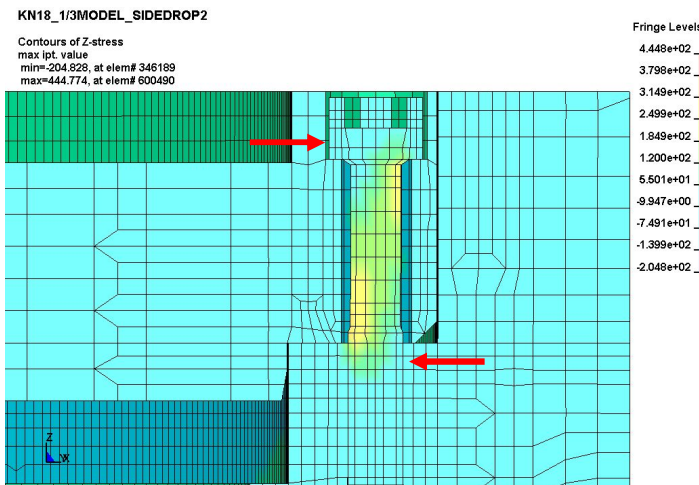
# 5. Numerical and Experimental Results

## [Drop behavior]

- ◆ IMPACT BEHAVIOR : 9m side drop
  - (▽) Shear behavior on the lid bolts nearest to the target
  - (▷) Deformation in the top and bottom impact limiter
  - (▷) Actual test results : outer welds of impact limiter near the target tore apart, and some of the wood layers extruded during the impact



[Deformation in the impact limiter]



[Shear behavior on the lid bolts]

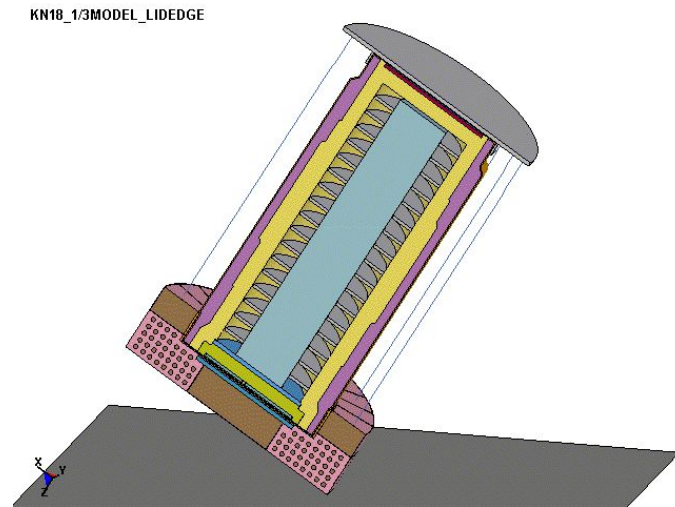


[Weld failure in the impact limiter]

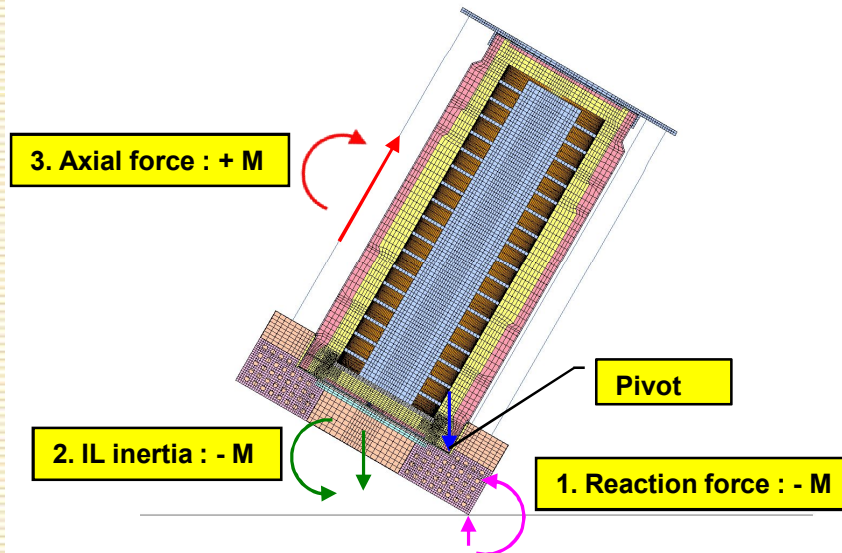
# 5. Numerical and Experimental Results

## [Drop behavior]

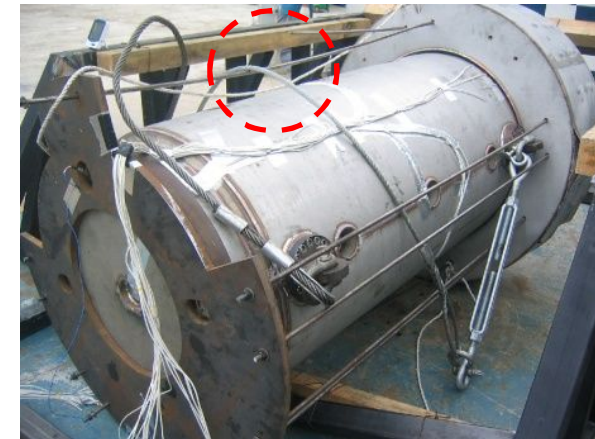
- ◆ **IMPACT BEHAVIOR : 9m lid edge drop**
  - (▽) **Drop orientation** : CG over the impact point  
- the angle of  $60^\circ$  from the cask axis to the target
  - (▽) **Loading on the impact limiter**
    - lid edge of the cask -> pivot
    - reaction force -> counter-clock wise M
    - IL own inertia force -> counter-clock wise M
    - tie-rod axial force -> clock wise M
    - > (▷) two of the 12 **tie-rod failed**
  - (▷) **Drop simulation results**



[Drop animation]



[Drop orientation]



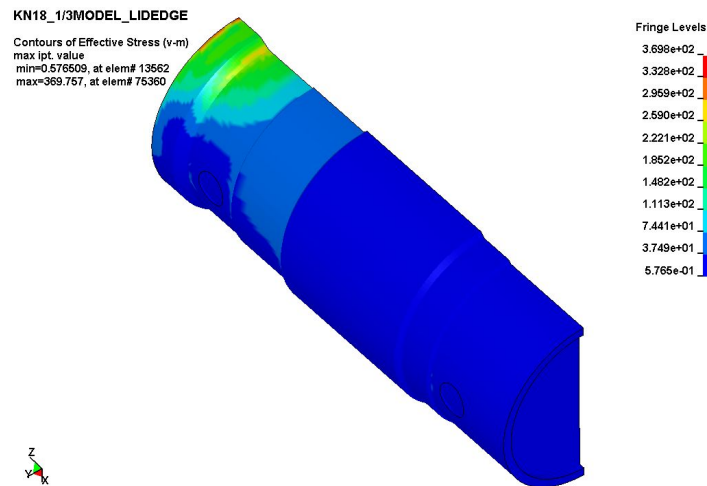
[Failure of the tie-rod]

# 5. Numerical and Experimental Results

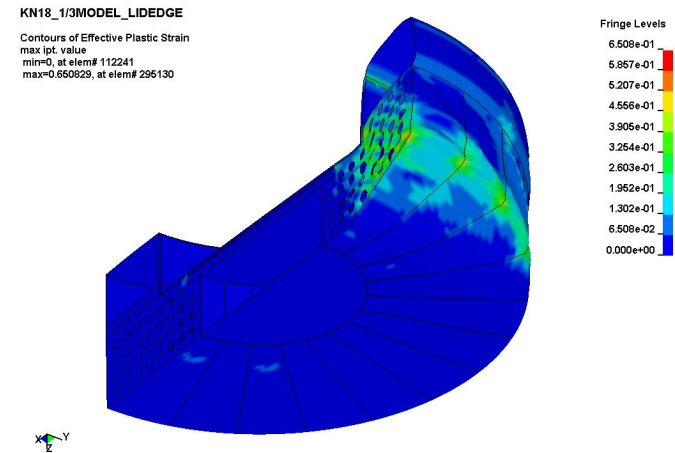
## [Drop behavior]

### ◆ IMPACT BEHAVIOR : 9m lid edge drop

- (▽) **Stress distribution** of the cask body
- (▷) **Plastic strains** in the top impact limiter
- **Fillet welds** of the impact limiter were assumed not to be failed in the analysis
- > (▷) **some welds failed** during impact in the actual test
- > analytical assumption : **conservative** prediction of cask stress



[Stress distribution of the cask body]



[Plastic strains in the top impact limiter]



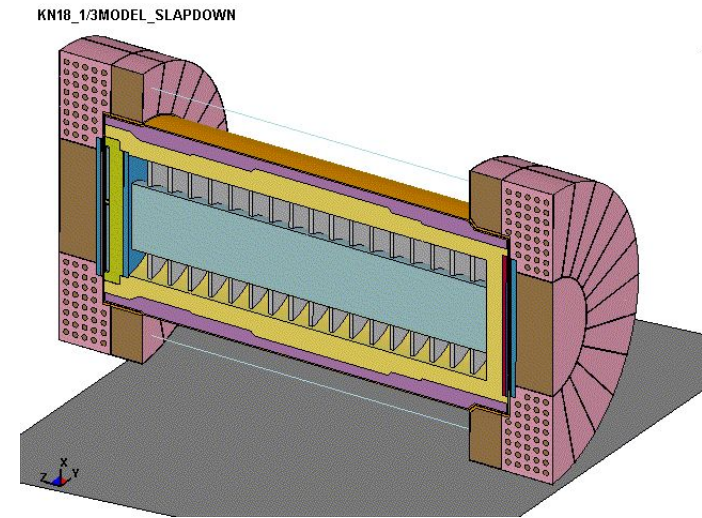
[Deformation in the impact limiter]



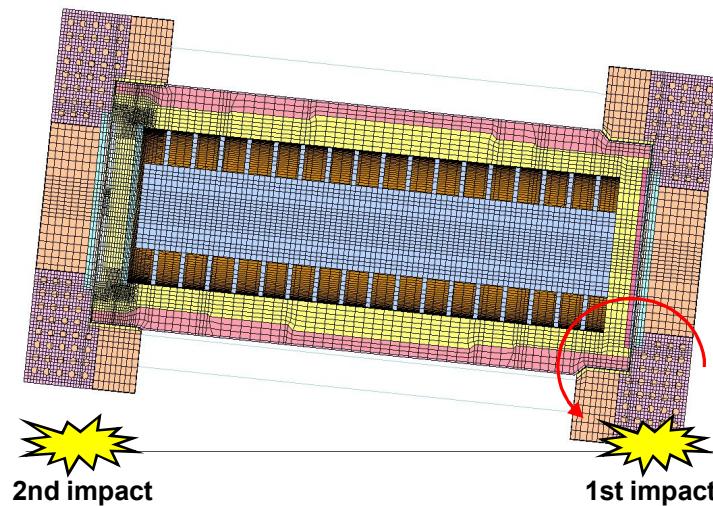
# 5. Numerical and Experimental Results

## [Drop behavior]

- ◆ **IMPACT BEHAVIOR : 9m slap down drop**
  - (▽) **Drop orientation** : CG over the impact point (X)
    - an angle of  $6^\circ$  from the cask axis to the target
    - after 1<sup>st</sup> impact, cask rotated under gravity
    - > 2 impact, 2<sup>nd</sup> impact > 1<sup>st</sup> impact : rebound
  - (▷) **Drop simulation results**
  - (▷) **Deformation** in the impact limiter

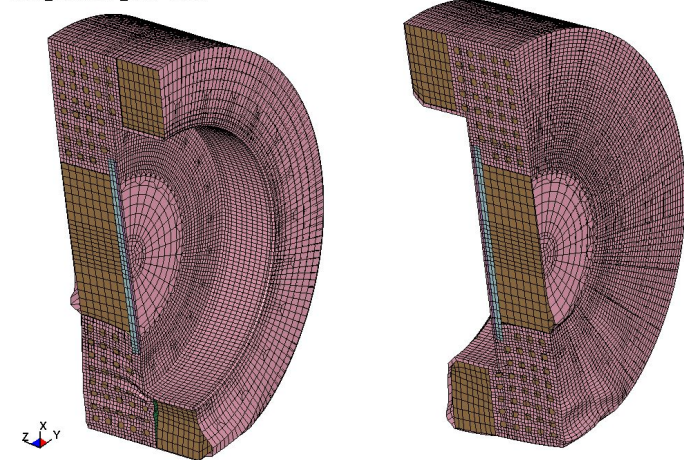


[Drop animation]



[Drop orientation]

KN18\_1/3MODEL\_SLAPDOWN

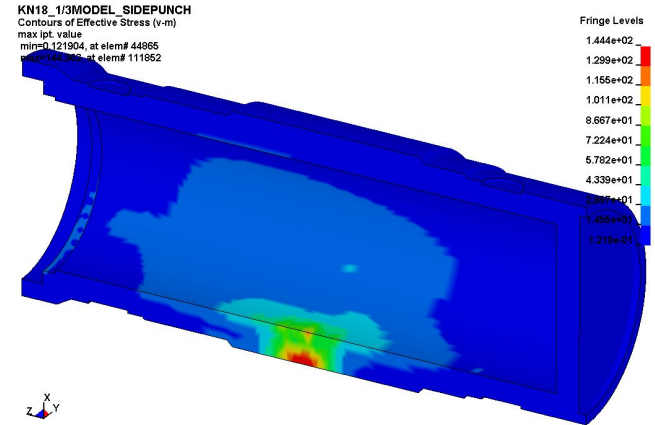


[Deformation in the impact limiter]

# 5. Numerical and Experimental Results

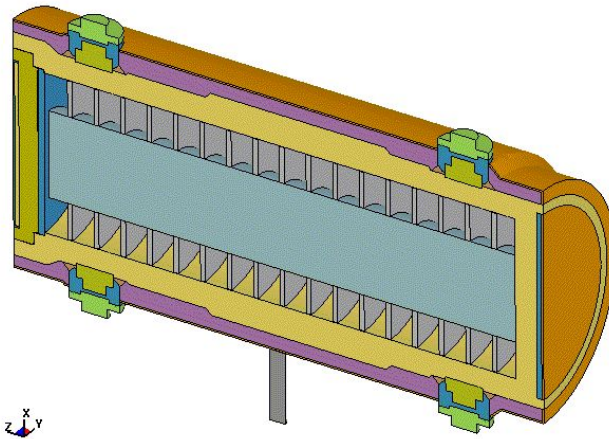
## [Drop behavior]

- ◆ **IMPACT BEHAVIOR : 1m side puncture**
  - (▽) **Drop orientation** : CG over the puncture bar
    - 1m free drop onto the puncture bar
  - (▽) **Drop simulation** results
  - **Cask deflected like a beam with a single support**
  - (▷) **Stress distribution** of the cask body
    - stress concentration near the impact point
  - (▷) **Punctured shape at the resin layer**
    - indentation of about 50mm in the resin layer



[Stress distribution of the cask body]

KN18\_1/3MODEL\_SIDE PUNCH



[Drop animation]



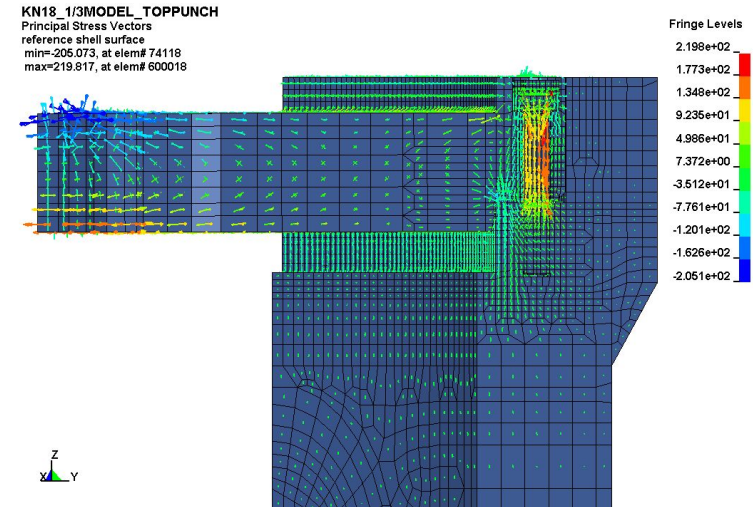
[Punctured shape at the resin layer]

# 5. Numerical and Experimental Results

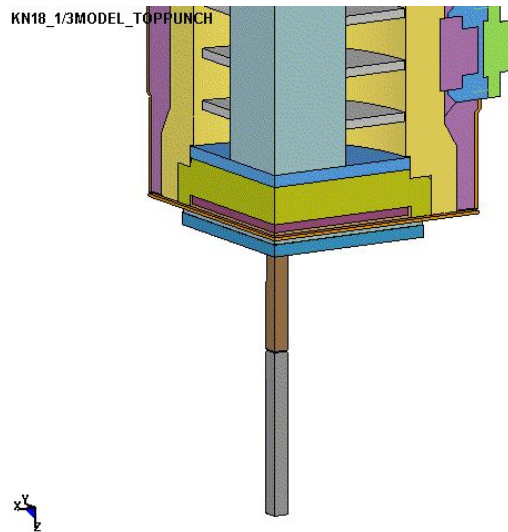
## [Drop behavior]

### ◆ IMPACT BEHAVIOR : 1m top puncture

- (▽) Drop orientation : CG over the puncture bar
- (▽) Drop simulation results
- (▷) Compression load path at the lid part
- (▷) Punctured shape at the impact limiter
  - indentation of about 150mm, bent pin
  - the impact behavior was dominated by crushing of the wood



[Load path at the lid-body interface]



[Drop animation]



[Punctured shape at the impact limiter]

## 6. Validation of Numerical Method

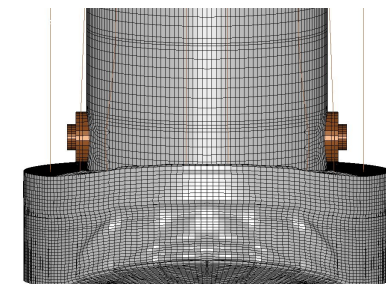
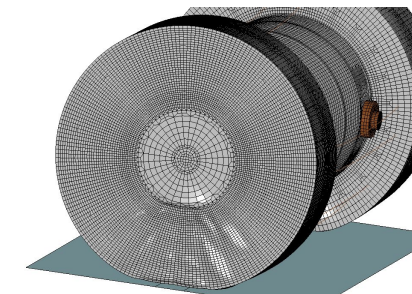
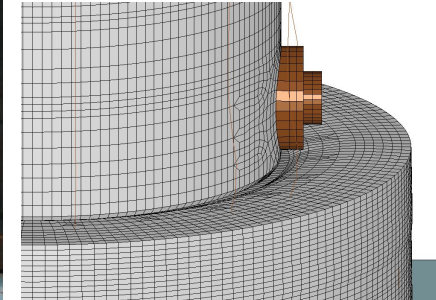
## [Comparison of results]

### ◆ OUTLINE OF VALIDATION

- Numerical results were compared with test results to verify the numerical simulation method used in the analyses
- Strain and acceleration measurements provide the essential components for the validation of the numerical methods
- Comparison items
  - deformation behavior
  - acceleration trace
  - strain trace

### ◆ DEFORMATION BEHAVIOR

- (▶) In general, there was good agreement between the analytical and test results
- But, some fillet welds of the impact limiter housing failed in the actual test
  - > conservative results



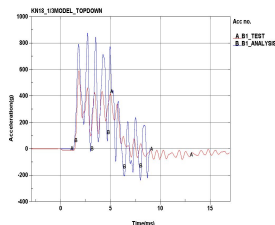
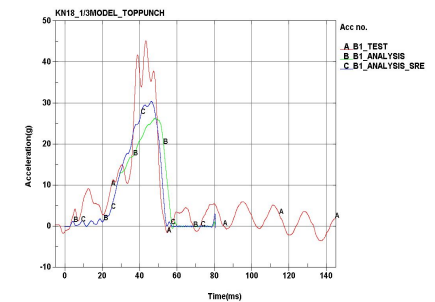
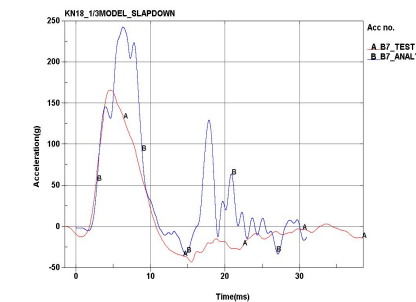
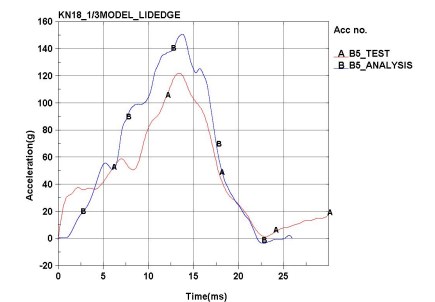
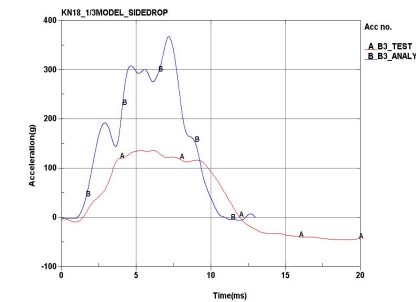
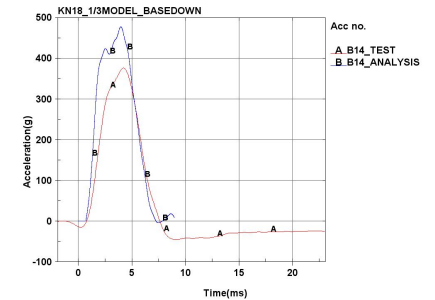
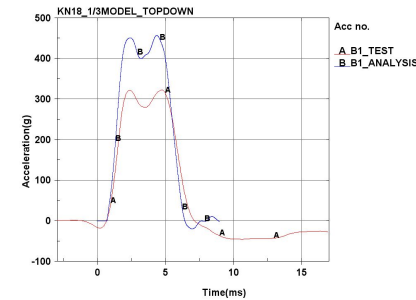
[Comparison of deformation behavior]

# 6. Validation of Numerical Method

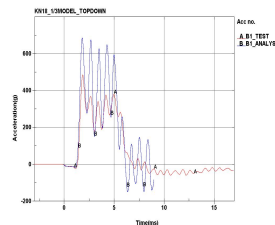
## [Comparison of results]

### ◆ ACCELERATION TRACE

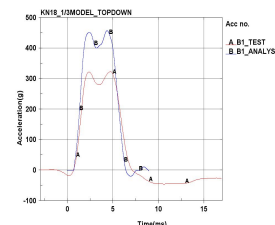
- Acceleration time histories were extracted from the FE model at the nodes nearest the accelerometers in the test
- Both data were filtered using same filtering method and same cut-off frequency
- (▽) Three different cut-off frequencies of 1650, 1000, 300Hz were applied
- (▷) Comparison of acceleration trace(300Hz)
- In general, acceleration traces from the analysis correspond very well with those from the test in terms of shape, magnitude, time scale
- Conservatively over-predict test results



[1650Hz]



[1000Hz]



[300Hz]

[Filtering of measurement data]

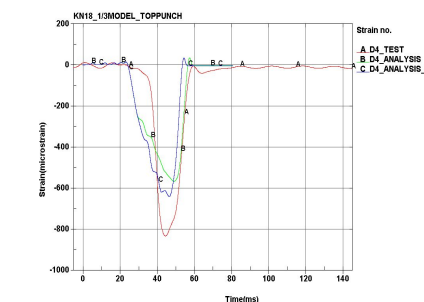
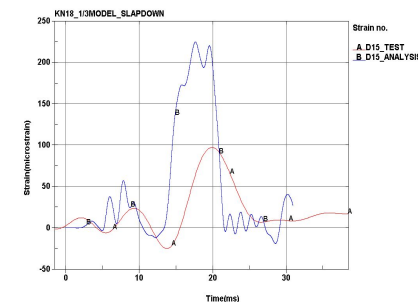
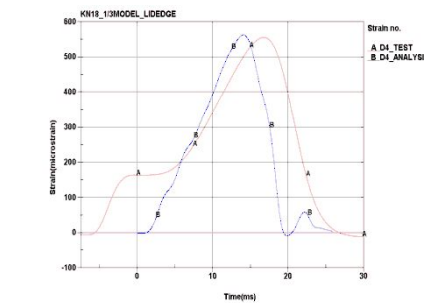
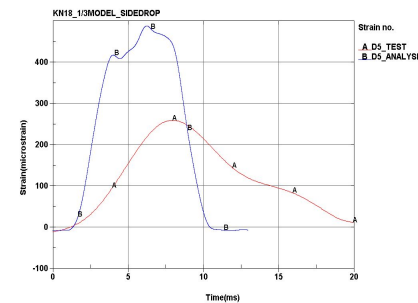
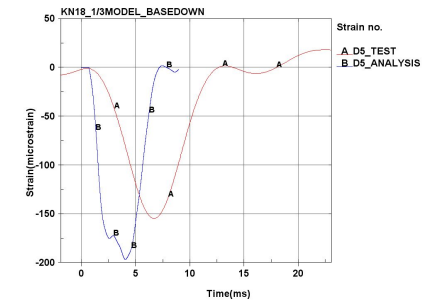
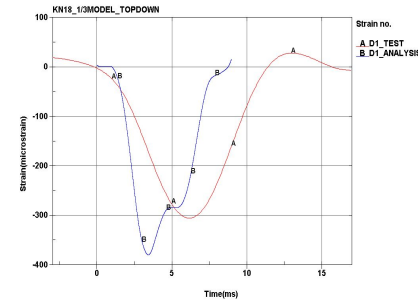
[Comparison of acceleration trace(300Hz)]

# 6. Validation of Numerical Method

## [Comparison of results]

### ◆ STRAIN TRACE

- **Strain** time histories were calculated from the relative distance between the nodes nearest the strain gauges in the test
- As in the acceleration data, both **data were filtered** using same method
- *Three different cut-off frequencies of 1650, 1000, 300Hz was used*
- (▶) Comparison of **strain trace(300Hz)**
- In general, as in the acceleration data, strain traces from the analysis also **correspond very well** with those from the test
- Conservatively **over-predict** test results



[Comparison of strain trace(300Hz)]

## 7. Summary and Conclusion

- ◆ In this study, a series of **actual drop tests** were performed using a **1/3-scale model** of newly developed **KN-18** SNF transport cask in Korea.
- ◆ In addition, **numerical simulations of the scale model** cask were performed for all the drop test conditions using **same numerical method** used in the safety analyses of the real cask.
- ◆ **Dynamic impact characteristics** of the KN-18 SNF transport cask under free-drop conditions have been investigated from the test and numerical simulation results.
- ◆ The numerical method used in the analyses have been **validated** through a **comparison of the test and numerical results**.  
In general, the numerical results are in **good agreement** with the test results.  
In addition, the numerical results consistently and conservatively **over-predicted** the test results for most of the evaluated cases.
- ◆ These good correlations with the drop test results demonstrate that the numerical simulation method used in the analyses of KN-18 SNF transport cask is **robust and reliable** in simulating and predicting the dynamic impact behavior of the cask.