

**ACCEPTABILITY OF DYNAMIC
FINITE ELEMENT ANALYSES –
MATERIAL FAILURE APPROACH**

Anindya Sen

Her Majesty's Inspector

Department for Transport (Central)

London, UK

Acceptable Failure Criteria

- Preamble
 - Analysis / assessment is robust and self-standing
 - Sufficient margin against failure available
 - Persons involved are well experienced in the area
 - Good trail of QA documentation available

Acceptable Failure Criteria

- Introduction
 - To elicit a “strain” based assessment criterion
 - Simple representative UN Type B package
 - “Accident” condition of transport – Para’ 727 (a) of TS-R-1 2009
 - 9 m “drop” on an unyielding “target” “so as to suffer maximum damage”
 - No follow-on fire test envisaged

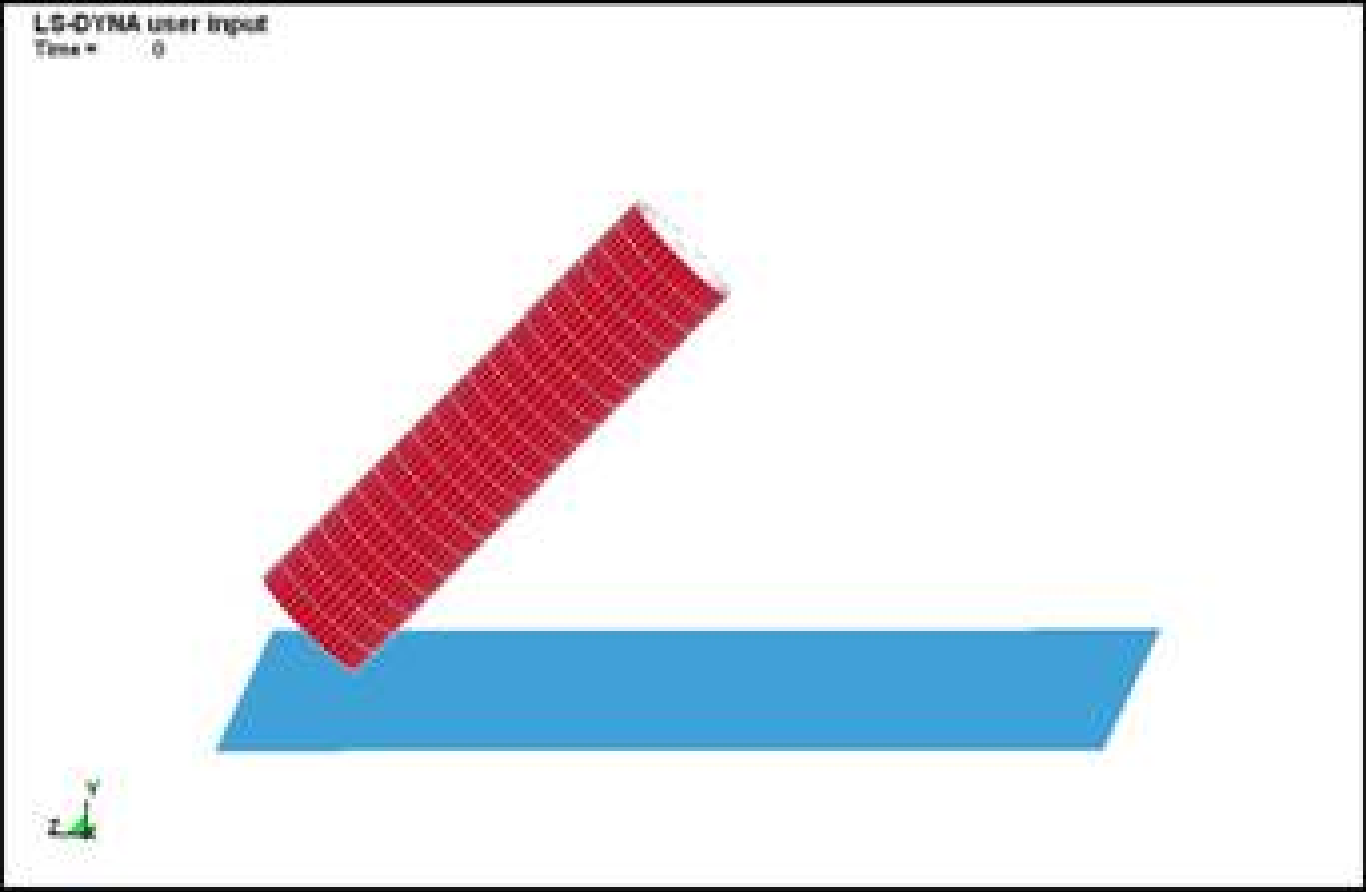
Acceptable Failure Criteria

- System Description
 - Thin walled hollow end-capped cylinder of typical Carbon Steel
 - 300 mm OD / 1200 mm Length / 10 mm wall thickness
 - Mass and Content (unspecified)
 - Calculated mass of ~95 kg
 - Simulated content of 13,754 N (in force, see later slide)
 - Drop orientation
 - Package longitudinal axis at 45° to horizontal for ease of modelling

Acceptable Failure Criteria

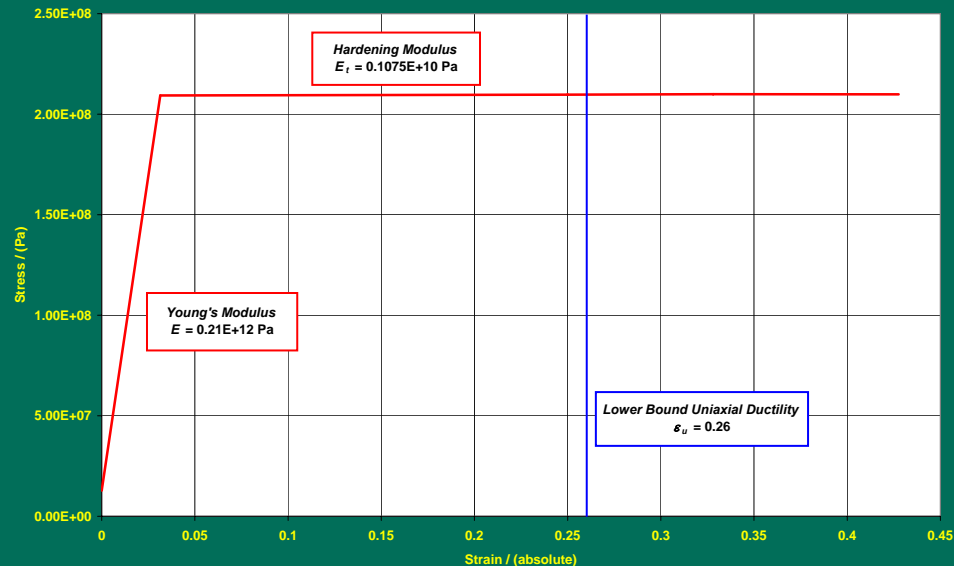
- Finite Element Model
 - Linear “Shell 163” element has been used for the “cylinder” as well as the “rigid” target (1400 elements and 1402 nodes)
 - 3 integration points through thickness (default)
 - General contact definition (“node to surface”)
 - Static and Dynamic Friction
 - $\sim 13 \text{ ms}^{-1}$ terminal velocity before impact
 - 0.175 second overall duration of simulation
 - No welds have been modelled
 - Default tolerance values
 - Package content simulated as 1kgf (9.81 N) vertical force at each node

Acceptable Failure Criteria



Acceptable Failure Criteria

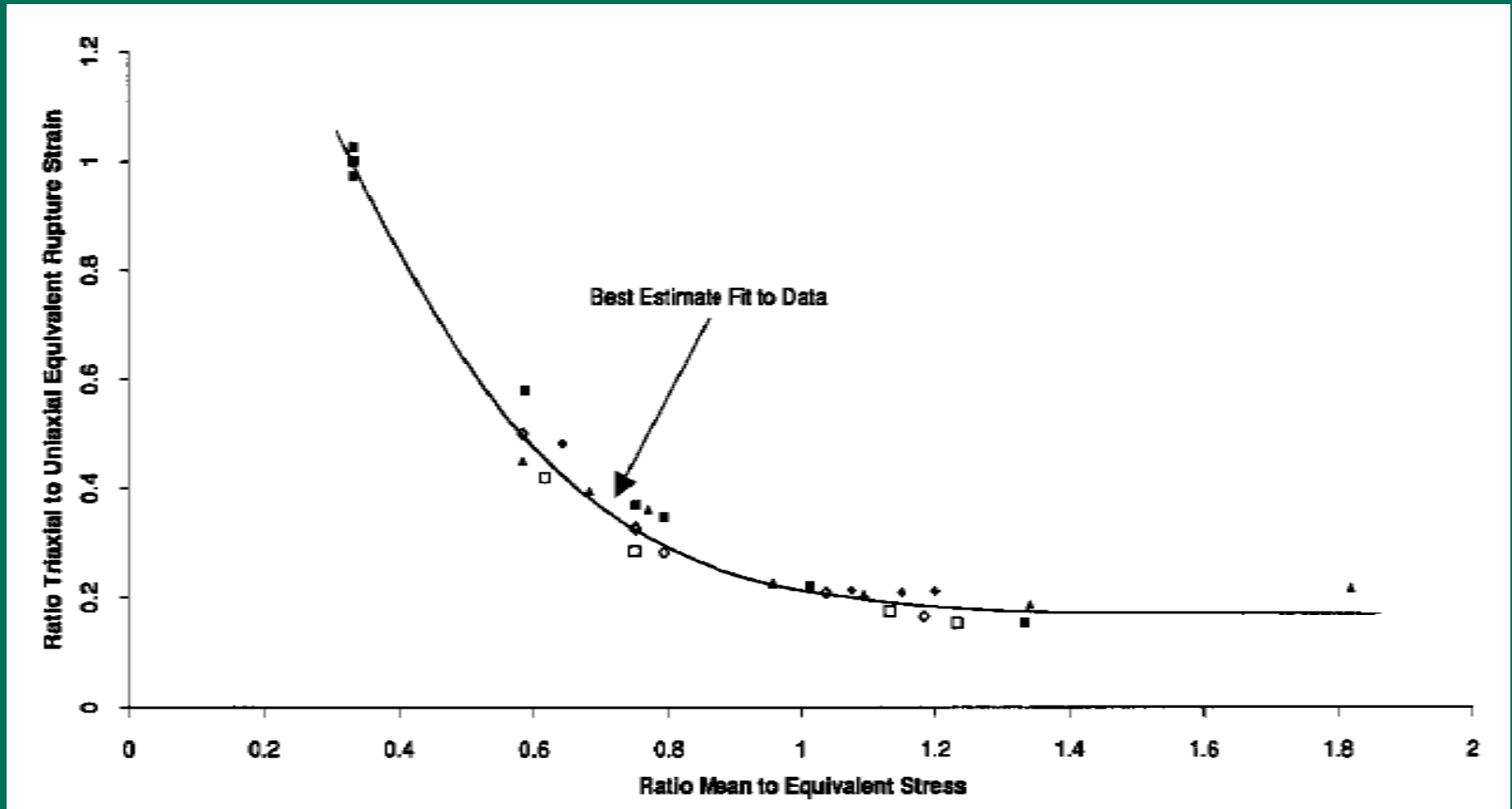
- Material Data (Physical & Tensile Properties)
 - Typical Carbon Steel from BS1501:Part 1 1980
 - Elastic – Plastic with Bilinear Hardening (**Mat_Plastic_Kinematic*)



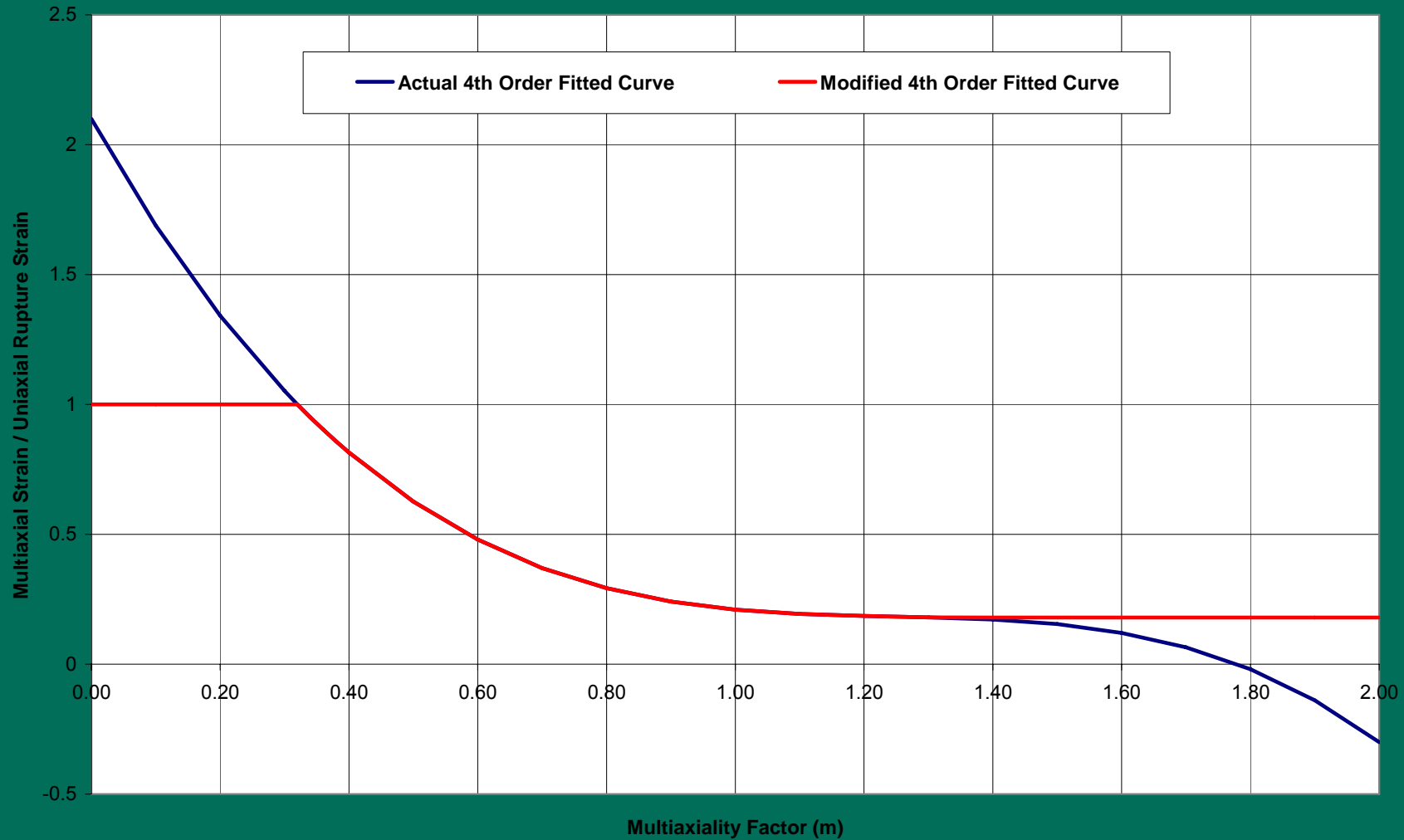
Acceptable Failure Criteria

- Multi-axial stress state “Strain – based” criterion – an example
 - R3 Impact Assessment Procedure – “strain (and energy) acceptability”
 - Predominant “plastic” response – “strain-based”
 - Multiaxial Stress State: rupture strain $\varepsilon_r \neq \varepsilon_u$ (uniaxial ductility or “elongation”)
 - $\varepsilon_r = f(m)$
 - Multiaxiality factor $m = \sigma_h / \sigma_{VM}$
 - $\sigma_h = - \text{trace}(\sigma_{kk}) / 3$, where σ_{kk} is the principal stress tensor
 - $\sigma_{VM} = \sqrt{(3/2 \sigma'_{ij} \sigma'_{ij})}$, where σ'_{ij} is the deviatoric stress tensor

Acceptable Failure Criteria

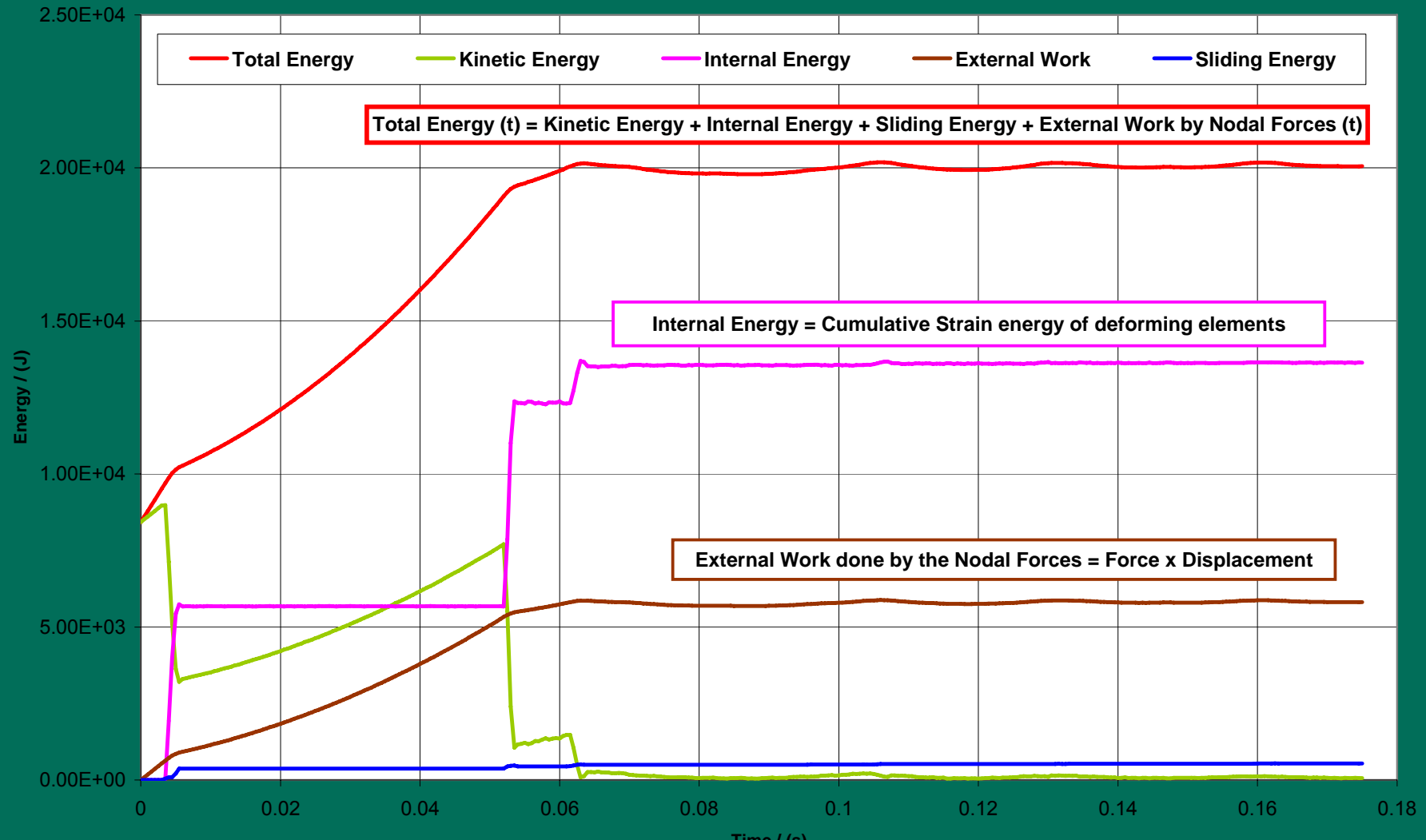


Acceptable Failure Criteria



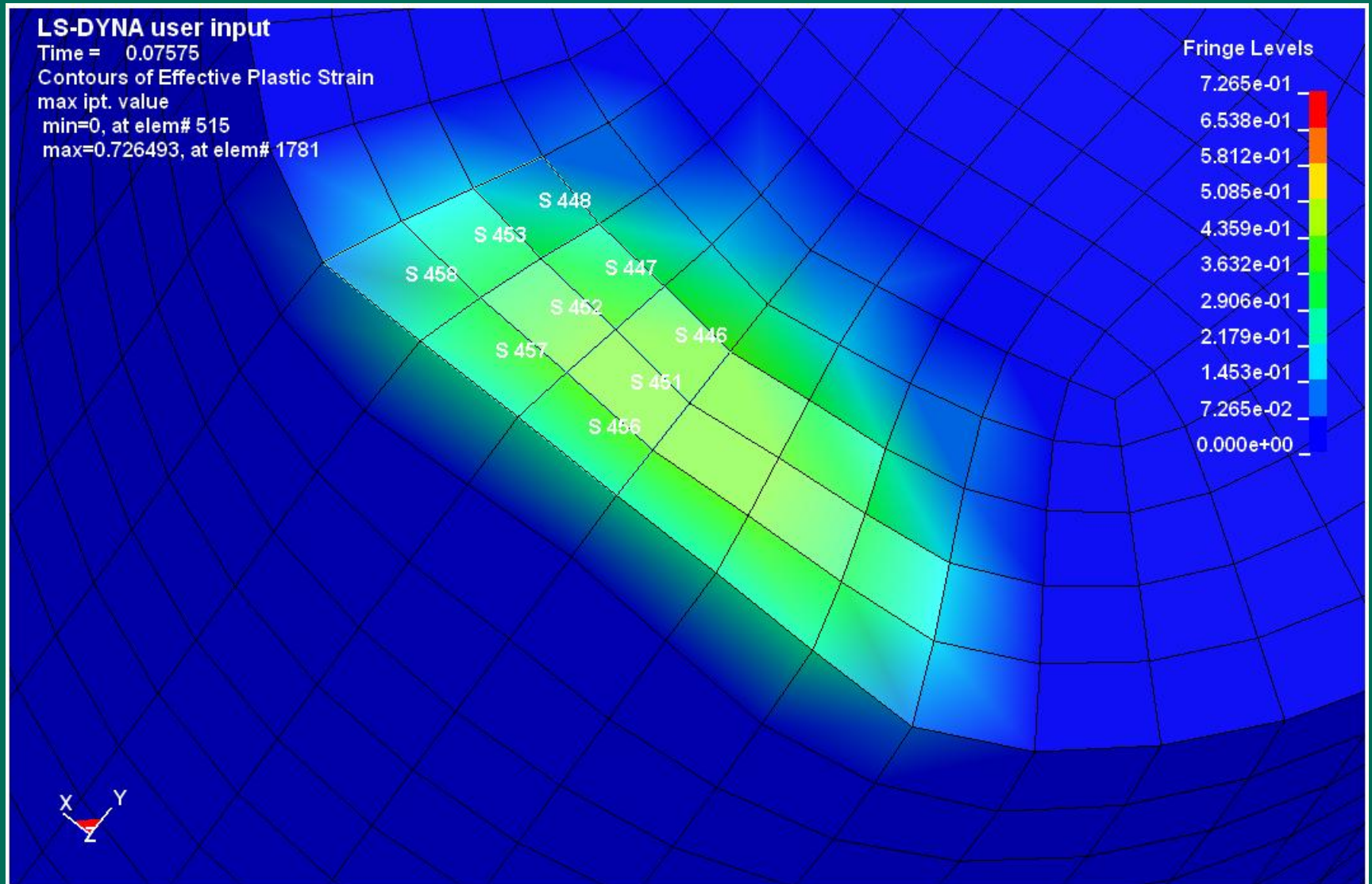
Acceptable Failure Criteria

- Energy Histories



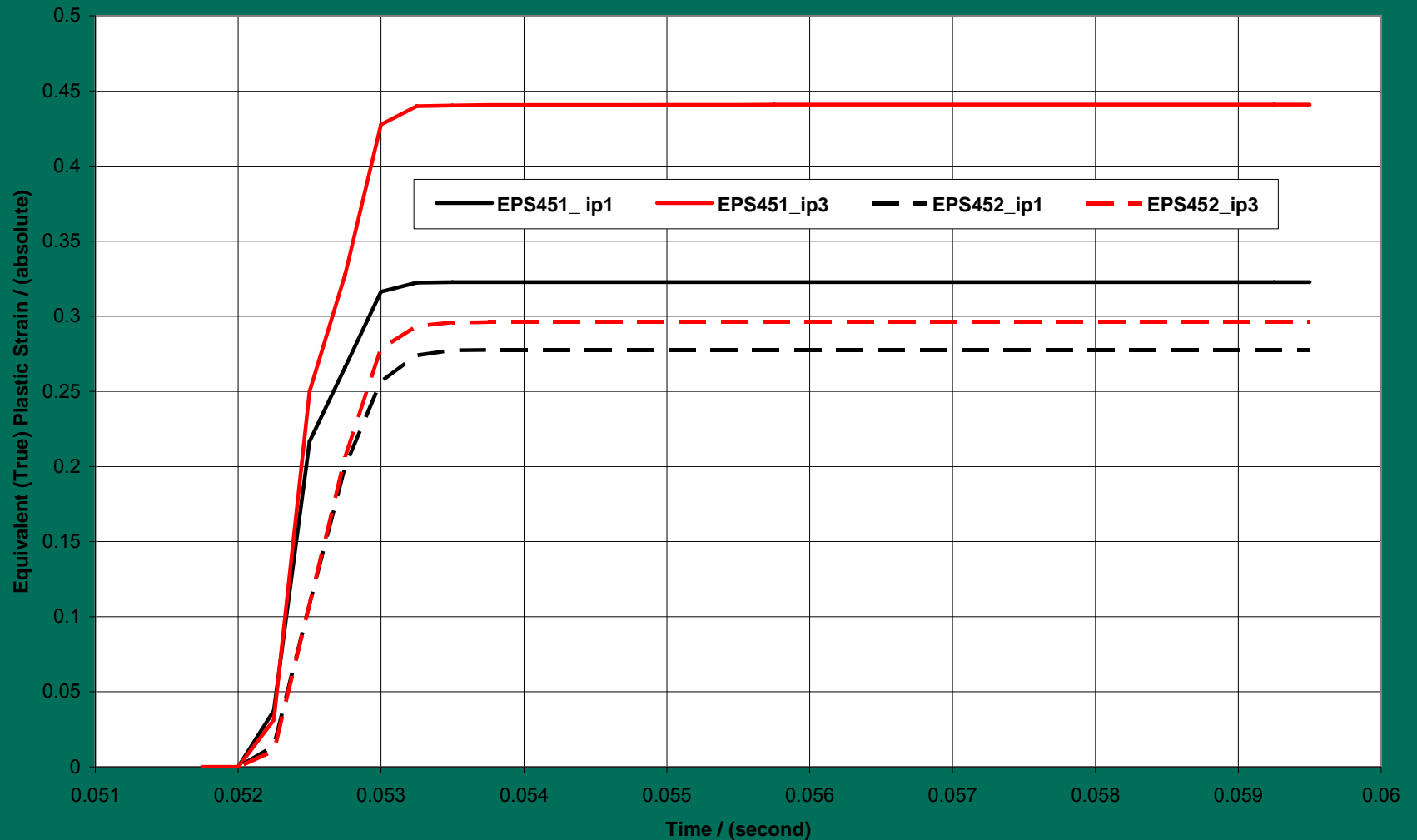
Acceptable Failure Criteria

- EPS



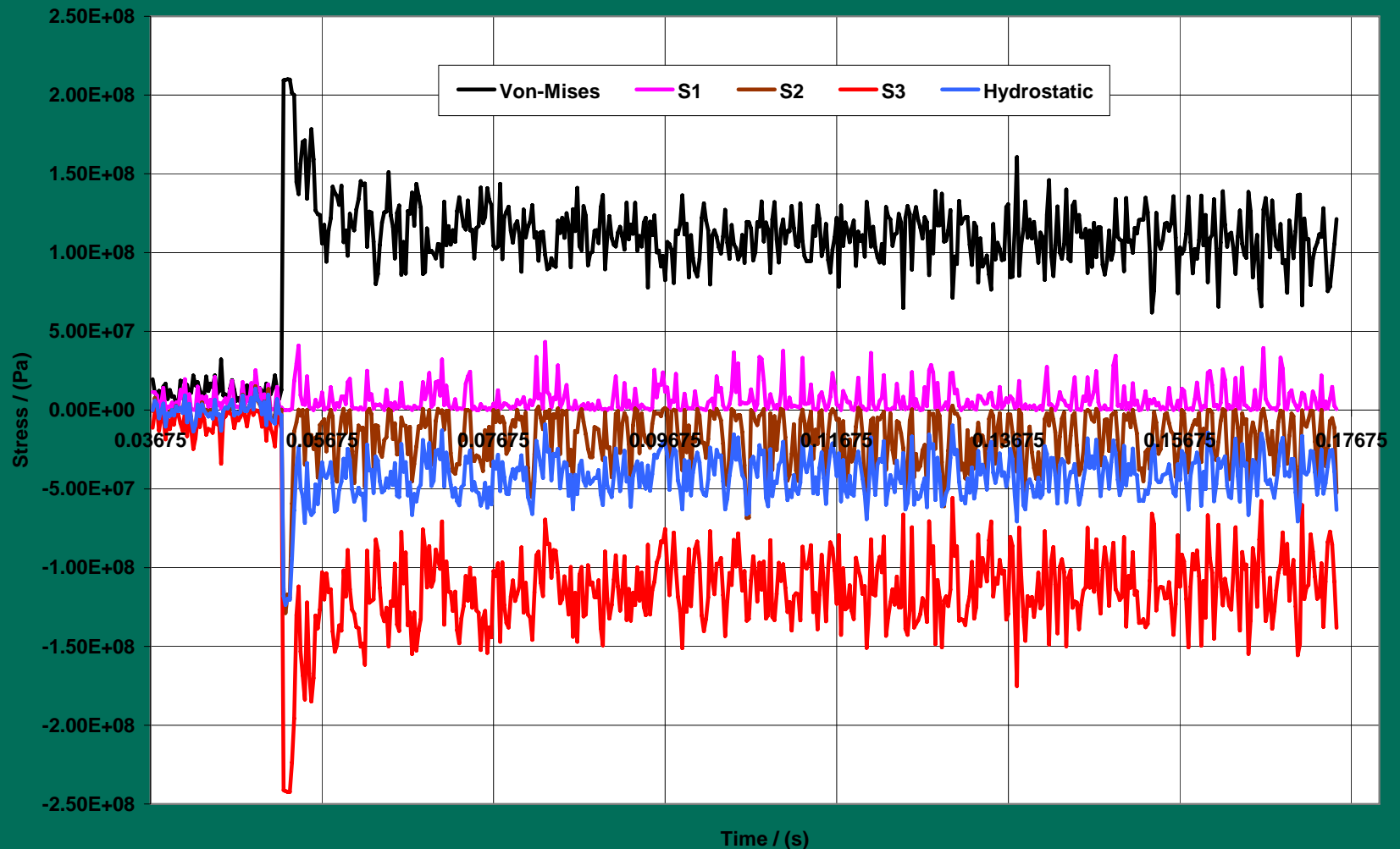
Acceptable Failure Criteria

- EPS history for E451 & E452



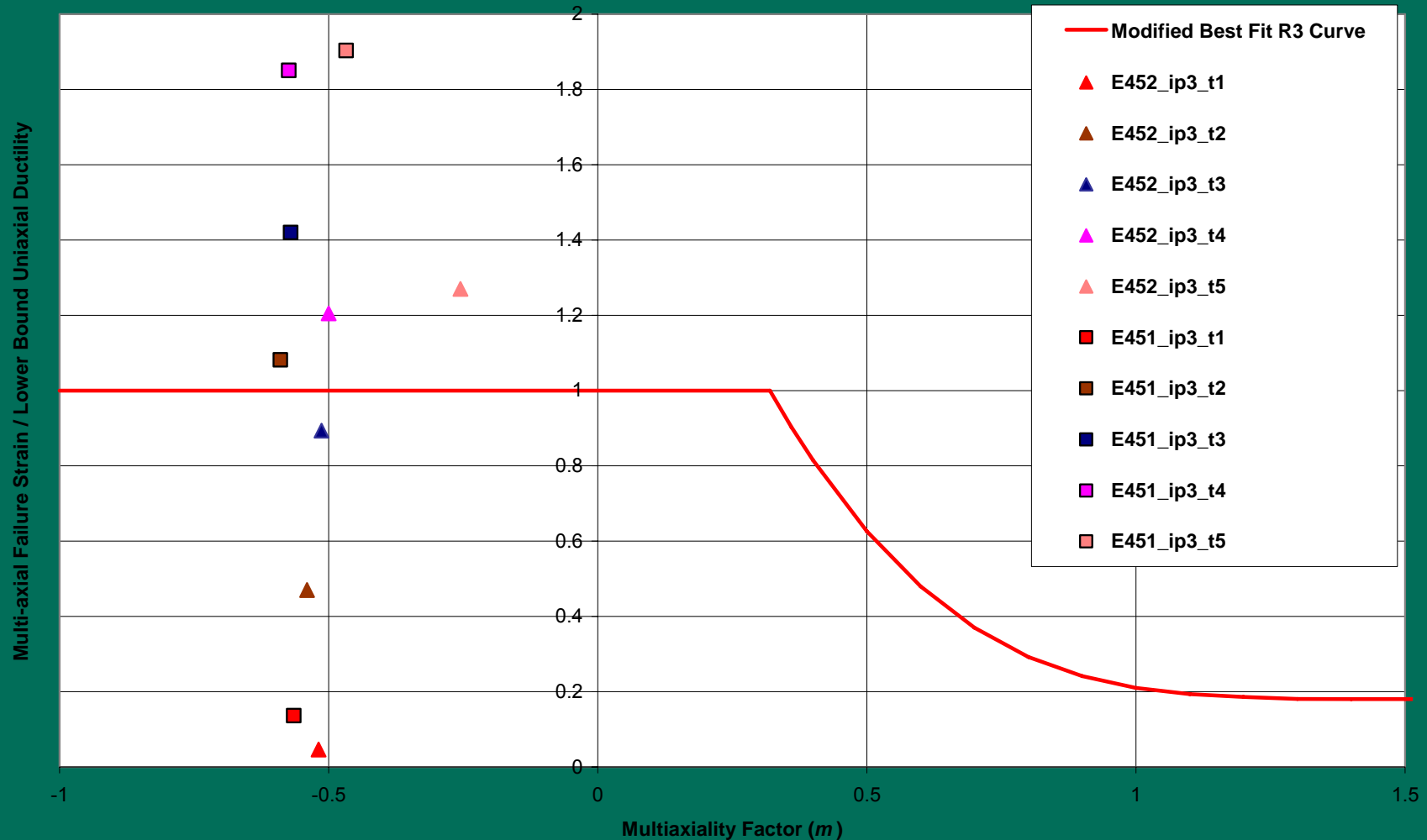
Acceptable Failure Criteria

- Stress histories for E451 (ip#3)



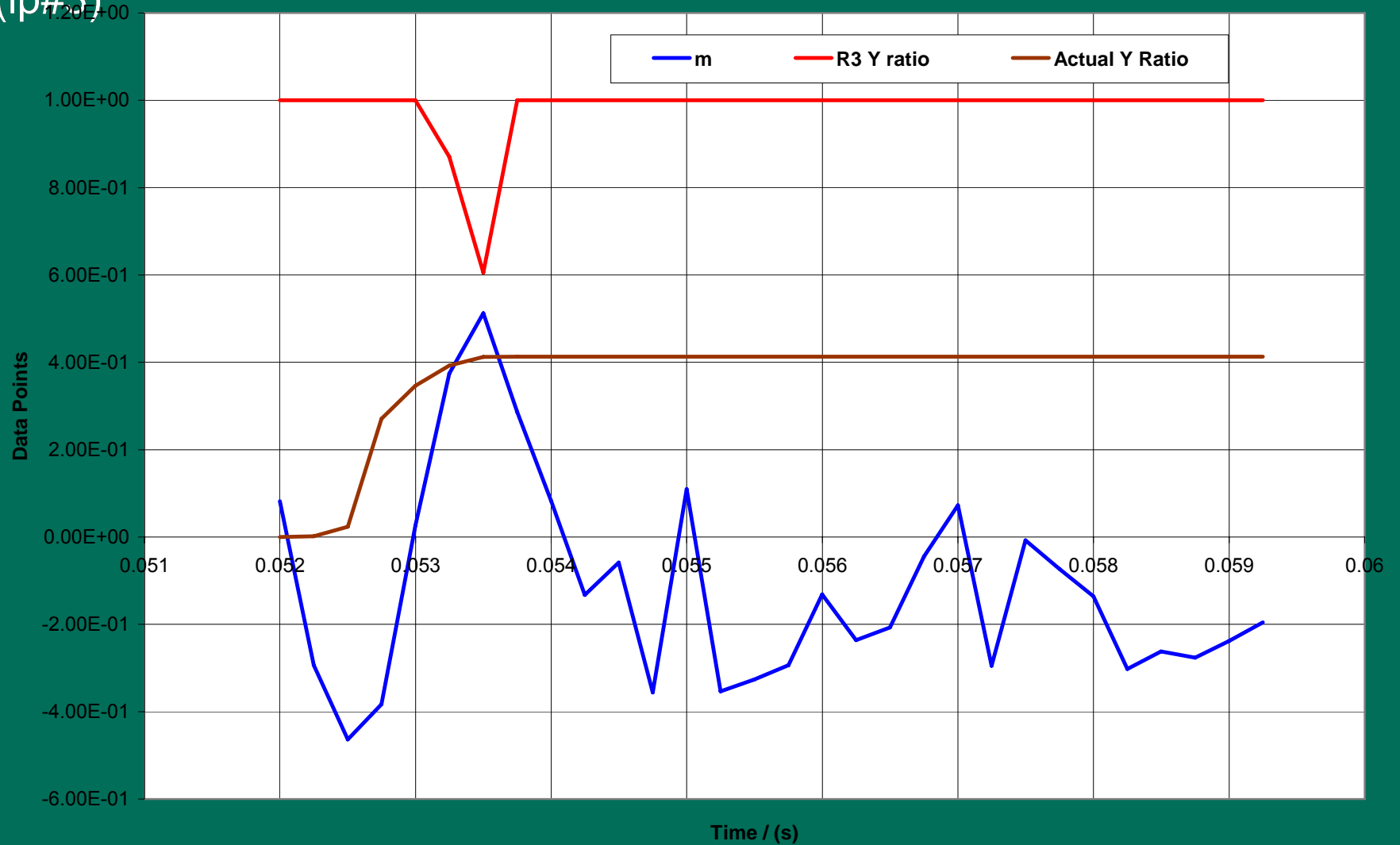
Acceptable Failure Criteria

- R3 Assessment for E451 (ip#3)



Acceptable Failure Criteria

- E441 (ip#3)



Acceptable Failure Criteria

- Conclusions
 - E451 failed due to very high accumulated plastic strain \gg lower bound ε_u
 - This procedure needs to be followed for each element with plastic strain (ε_{eqv}^p)
 - Pessimism
 - Lower bound material properties
 - Bilinear hardening curve (e.g. strain rate $\sim 400 \text{ s}^{-1}$ “high” – significant work hardening ignored)
 - No shock absorbing material used
 - Future work planned
 - Appropriate factor of safety (FOS) against allowable multiaxial failure strain
 - Compare with ASME VIII Division 2 Section 5.3 (“Protection Against Local Failure”)

Acceptable Failure Criteria

- Questions if any please